

# **KISSsoft 03/2015 – Instructions 072**

## **Contact Analysis in the Cylindrical Gear Calculation**

Ho 06.05.11, Update 11.10.2011 Ho; Update 16.09.2013 Ho; Update 31.07.2015 Ho

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# 1 Initial Situation

Since the 2011 release it is now possible to include and take into account shaft deformation directly in cylindrical gear analysis. If you do so, the deformation that occurs is determined on the basis of the shaft dimensions and included directly in the cylindrical gear analysis. To illustrate this new functionality, KISSsoft is providing a number of pre-defined example analyses for cylindrical gears and the shafts associated with them. The relevant examples are highlighted in the figure below.

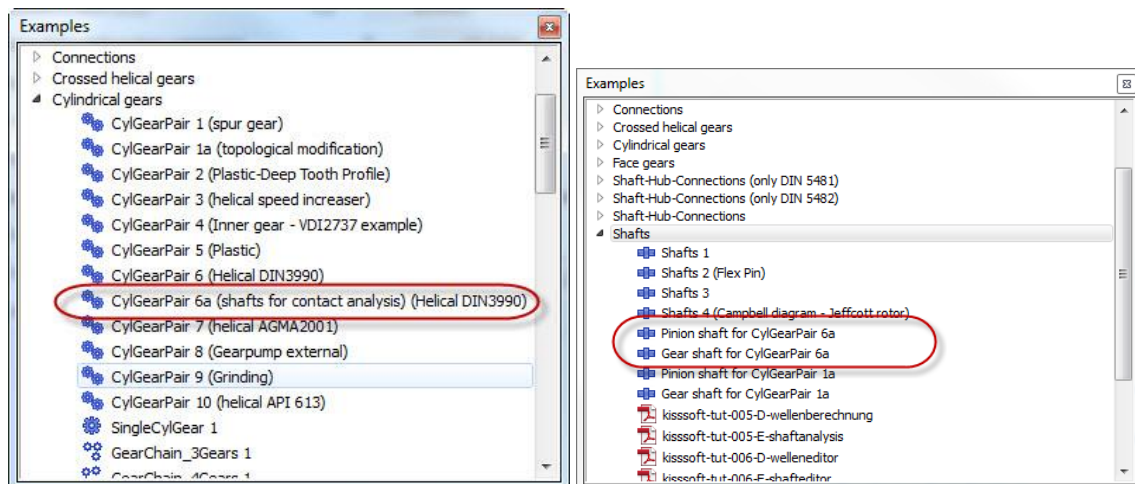


Figure 1. pre-defined calculation files for cylindrical gear pair and shaft analysis

## 1.1 Work steps

Work through the following steps with these examples:

Step 1: Analyze the current situation according to the relevant standard

Step 2: Analyze the current situation, taking into account the shaft deformation, with or without contact analysis

Step 3: Determine the necessary modifications for the tooth trace on the pinion and the gear

Step 4: Include the modifications in the cylindrical gear calculation

Step 5: Analyze the optimized situation, taking into account the shaft deformation

## 2 Solution

### 2.1 General notes

These work steps are designed to illustrate the basic procedure, to better identify and describe this new functionality.

The face load factors take into account the effect of unequal load distribution across facewidth on flank pressure  $K_{H\beta}$ , on tooth root stress  $K_{F\beta}$  and on scuffing stress  $K_{B\beta}$ . The KISSsoft system has a number of different ways in which you can input  $K_{H\beta}$ :

- Input  $K_{H\beta}$  directly, method A;
- Input deformation  $f_{sh}$  and manufacture tolerance  $f_{ma}$ : The KISSsoft shaft analysis functions can calculate the exact flank line variation due to deformation (torsion and bending) in the plane of action, method B;
- An approximate analysis according to ISO6336 (or DIN 3990). To do this, input the bearing distance  $l$ , the distance  $s$  of the pinion shaft, and the outside diameter of the pinion shaft, method C.

#### 2.1.1 Step 1: Analyze the current situation

Load the current cylindrical gear calculation file CylGearPair 6a.Z12 into the cylindrical gear pair module. Compare the data inputs for the face load factor with the current dimensions taken from the shaft analysis of the pinion. Change the settings and compare the determined face load factors for  $K_{H\beta}$  with the subsequent analysis results from steps 2 through 5.

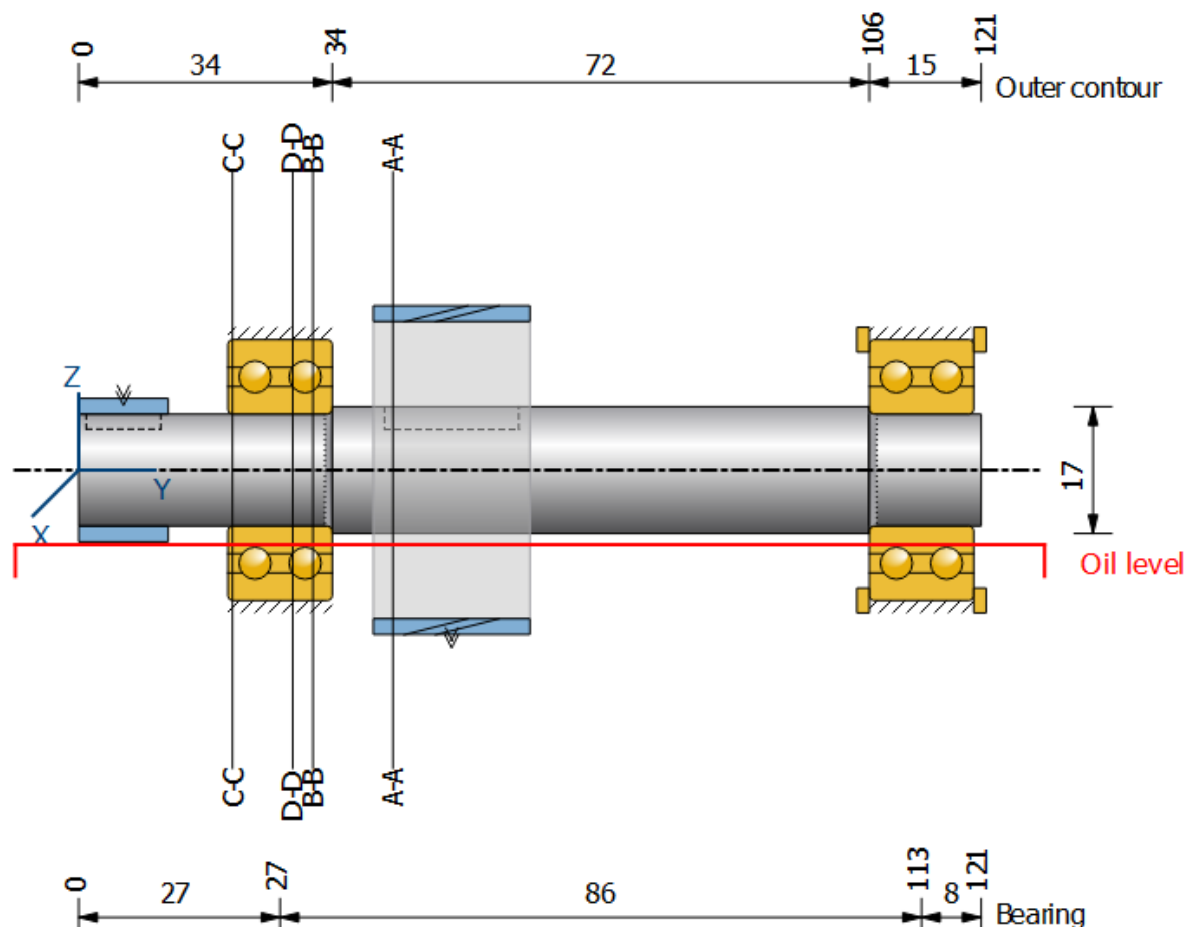


Figure 2. Pinion shaft dimensions and support

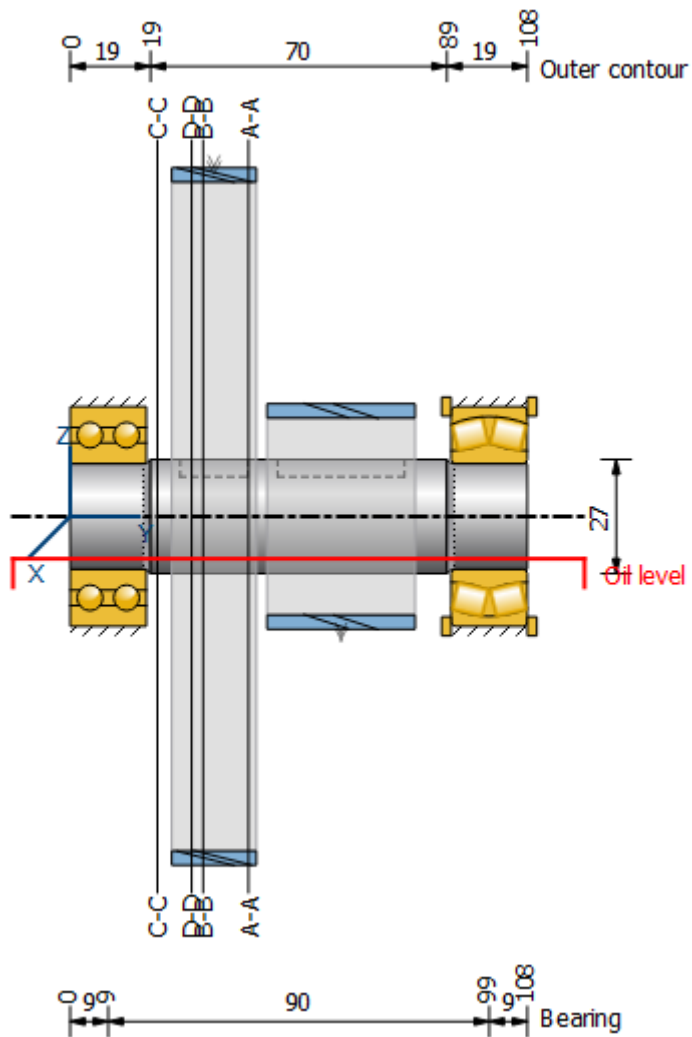


Figure 3. Gear shaft dimensions and support

To define the inputs for the face load factor, go to “Factors” tab. In the “Face load factor” section, first select “Calculation according calculation method” in the drop down list. Then click in the plus button on the right of the section to define more inputs for the load face factor.



Figure 4. Defining the face load factor

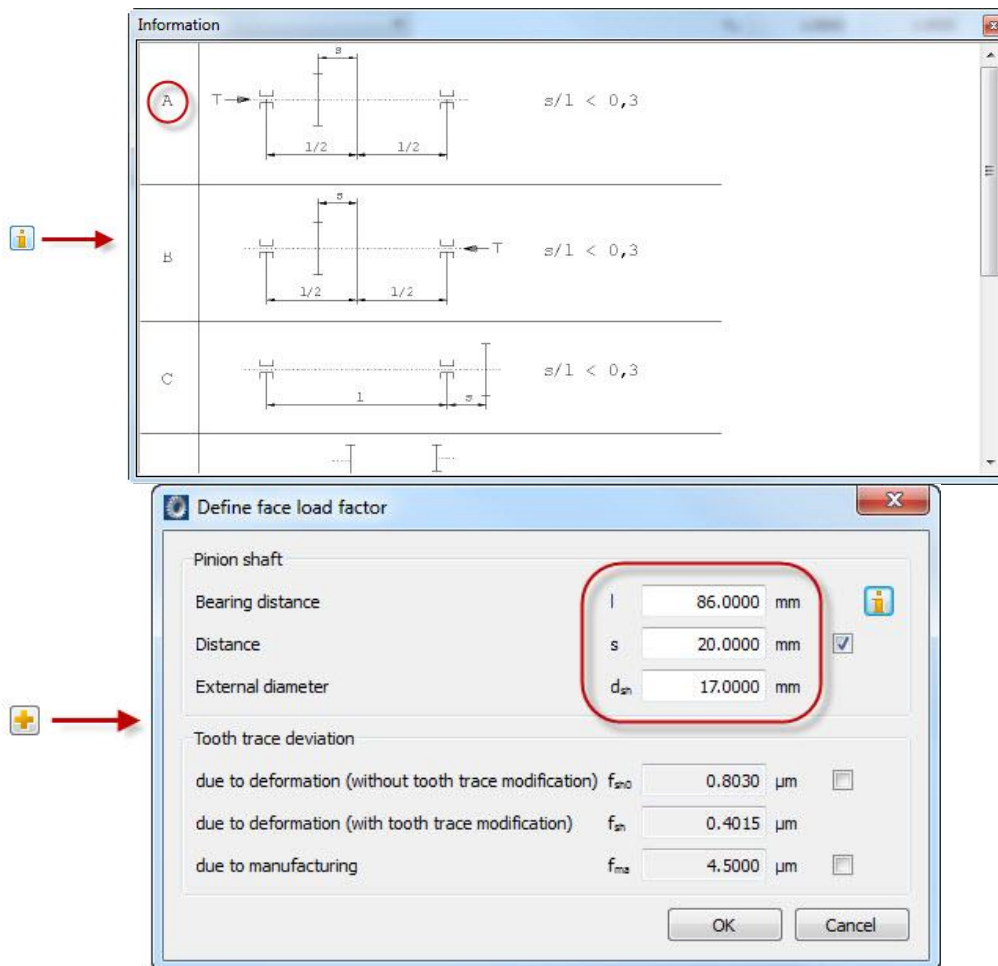


Figure 5. Inputs that take into account pinion dimensions (for method C)

The application factor  $K_A$  must be set to 1 so that it can be compared with the results of the subsequent analysis at a later point in time

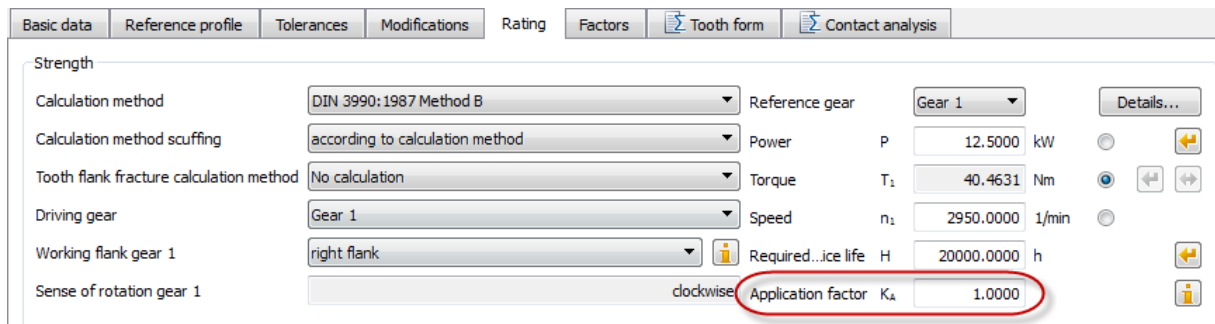


Figure 6. Application factor

The following message appears when you run this calculation.

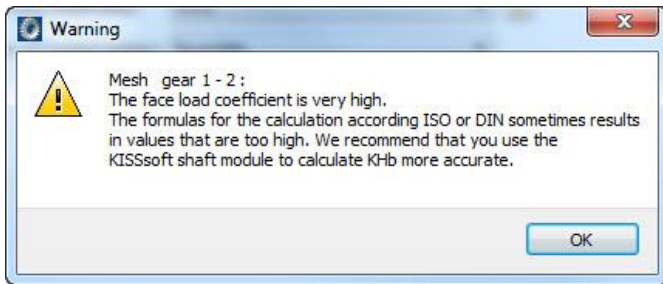


Figure 7. Warning message stating that the face load coefficient is too high

**Note:**

Any face load factor calculated according to method C which have a value greater than 1.5 must be checked in more detail. Method C is often very conservative. In other words, the calculated factor is too high and therefore the calculated safeties are too low. The factor should therefore be calculated in greater detail. You can do this by verifying the value more precisely. To do this, use the KISSsoft Shaft calculation module or the shaft analysis functions integrated in contact analysis in the KISSsoft cylindrical gear calculation functionality.

	Gear 1	Gear 2
Contact ratio (Transverse/Overlap/Total)	1.568 / 1.027 / 2.595	
Actual tip circle (mm)	45.692	163.973
Root safety	2.761	2.106
Flank safety	1.021	0.817
Safety against scuffing (integral temperature)	4.777	
Safety against scuffing (flash temperature)	8.226	
Safety against micropitting (Method B)	0.518	

Figure 8. Calculation results according to the standard, method C, and taking into account the influence of pinion geometry

The result shows the calculation of  $K_{HB}$  with the appropriate settings according to the standard. This calculation only includes the dimensions of the pinion shaft. However, the gear shaft will also deform when placed under load, and this method cannot take this effect (which is usually significantly smaller) into account.

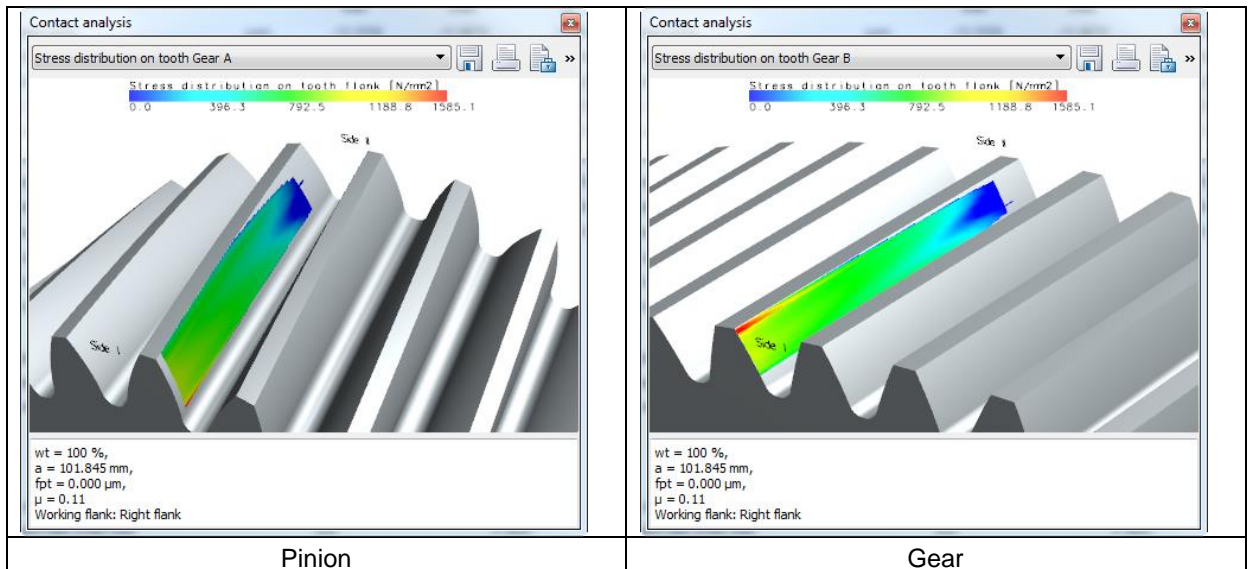


Figure 9. stress distribution calculated with  $K_{H\beta} = 2.2$

### 2.1.2 Step 2: Analyze the current situation, considering shaft deformation

This calculation step determines the face load factor using the contact analysis in the KISSsoft cylindrical gear calculation functionality. You can perform the calculation with a complete contact analysis. In this case, the calculation will take significantly longer to run and the result for  $K_{H\beta}$  is more accurate as it takes into account the shafts deformation.

The necessary shaft calculation files have already been assigned correctly in the CylGearPair 6a (...).Z12 cylindrical gear calculation file supplied with the system.

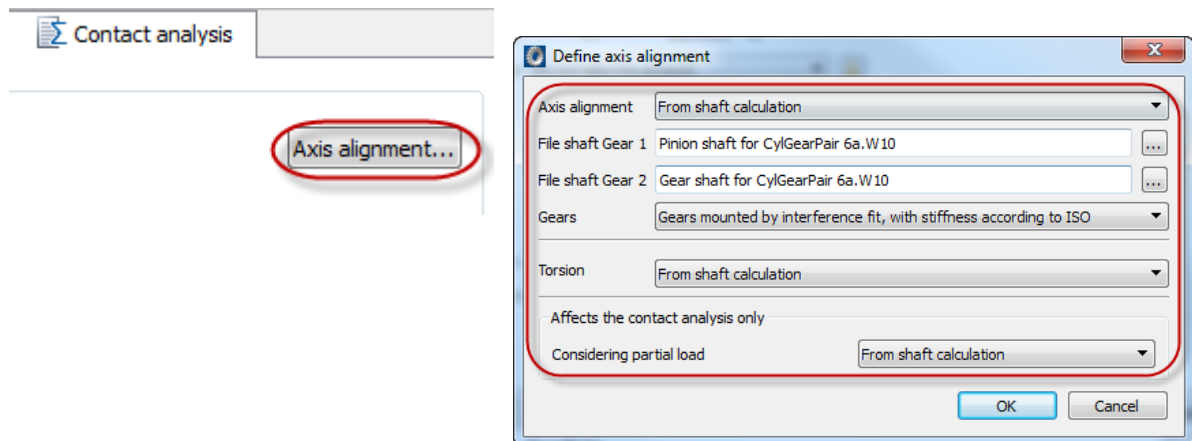


Figure 10. Determining  $K_{H\beta}$  with contact analysis

**Note:**

- a) If you want to consider tooth trace or profile corrections in the calculation, you must enable these modifications in the "Modifications" tab before running the contact analysis. If tooth trace corrections have been entered, you must enable them for this calculation step in the contact analysis. You do not need to enable profile corrections if only  $K_{H\beta}$  is to be calculated
- b) If the warning message does not appear, this means either that you have not input any modifications or that the modifications are already enabled.



To activate modifications in the "Modification" tab, enable the status as shown in the next figure.

Gear	Type of modification	Value [µm]	Factor 1	Factor 2	Status	Information
Gear 1	Tip relief, arc-like	5.0000	0.8651		active	dCa=44.4 mm
Gear 2	Tip relief, arc-like	5.0000	0.5000		active	dCa=163.4 mm

Figure 11. Enabling corrections in the "Modification" tab

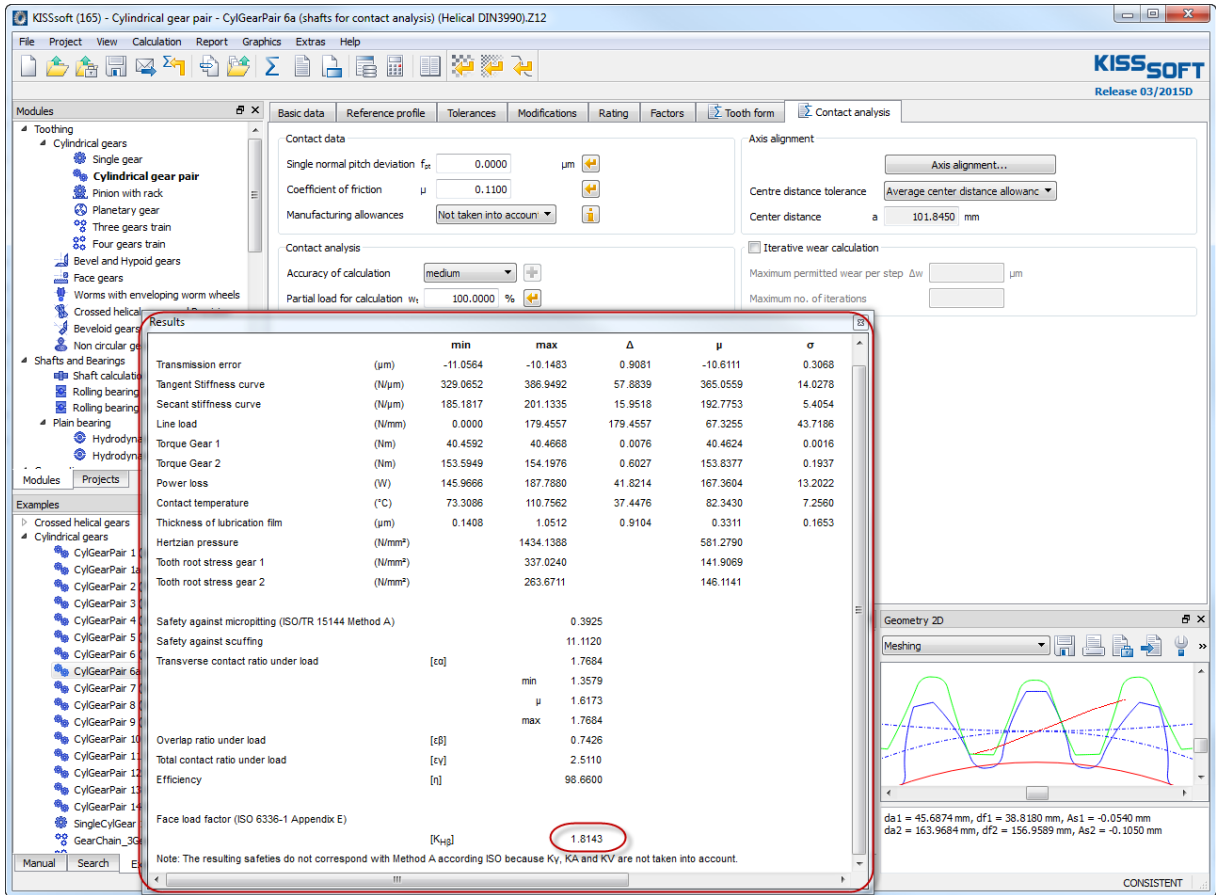


Figure 12. Results overview for a complete contact analysis without tooth trace modifications

In contrast to the calculation performed according to the equation in the standard, as shown in Step 1, the effective occurring face load factor changes to 1.81 in analysis step 2. This can now be used directly in the calculation. The safeties determined in the calculation then change accordingly.

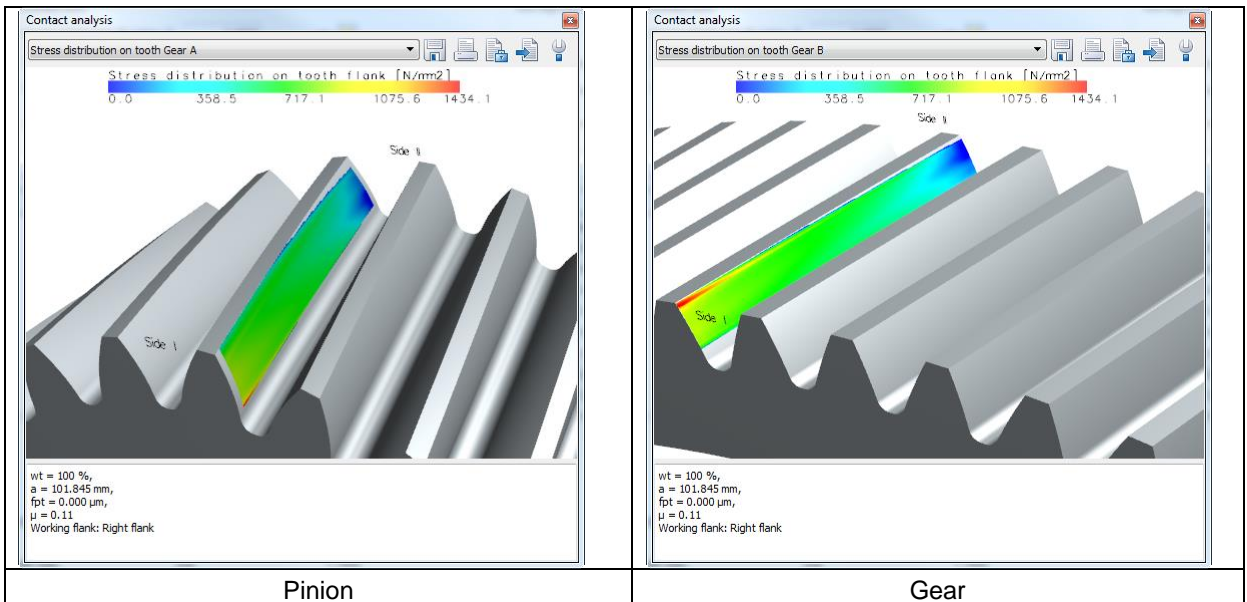


Figure 13. stress distribution calculated with  $K_{HB} = 1.81$

### 2.1.3 Step 3: Determine the necessary modifications for the tooth trace on the pinion and the gear

To do this, load the file 'Pinion shaft for CylGearPair 6a.W10' into the Shaft calculation module. Then open the "Tooth trace modification" tab as shown in the figure below.

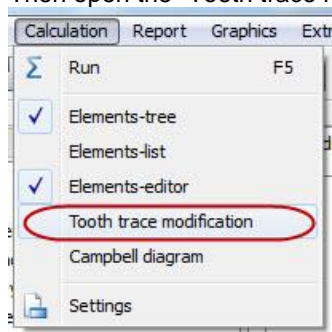


Figure 14. Calculating the tooth trace modification

Run the calculation after this. The next figure shows the results determined for the pinion. The calculation and modeling have been performed in such a way as to represent the counter gear as an idealized gear with infinite stiffness.

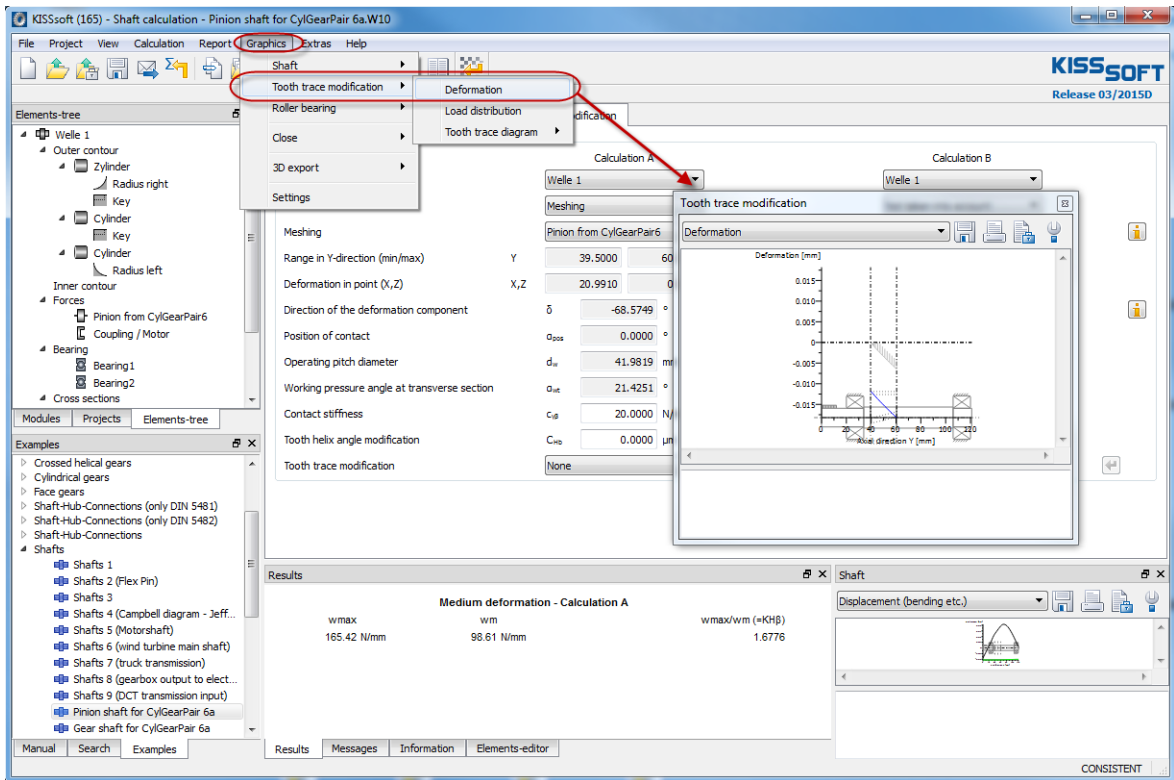


Figure 15. Results overview without tooth trace modification for the pinion

To reach an ideal modification, enter the helix angle modification  $C_{H\beta}$  step-by-step. You can then check the result, i.e. the correct input for the size and the sign. The deformation graphic shows an optimum proposed value (shown here in gray). If values have been defined for the helix angle modification  $C_{H\beta}$ , the graphic also shows the defined correction (shown here in green). The value for  $K_{H\beta}$  should be reduced (ideally to 1). The current value is then displayed in the results window.

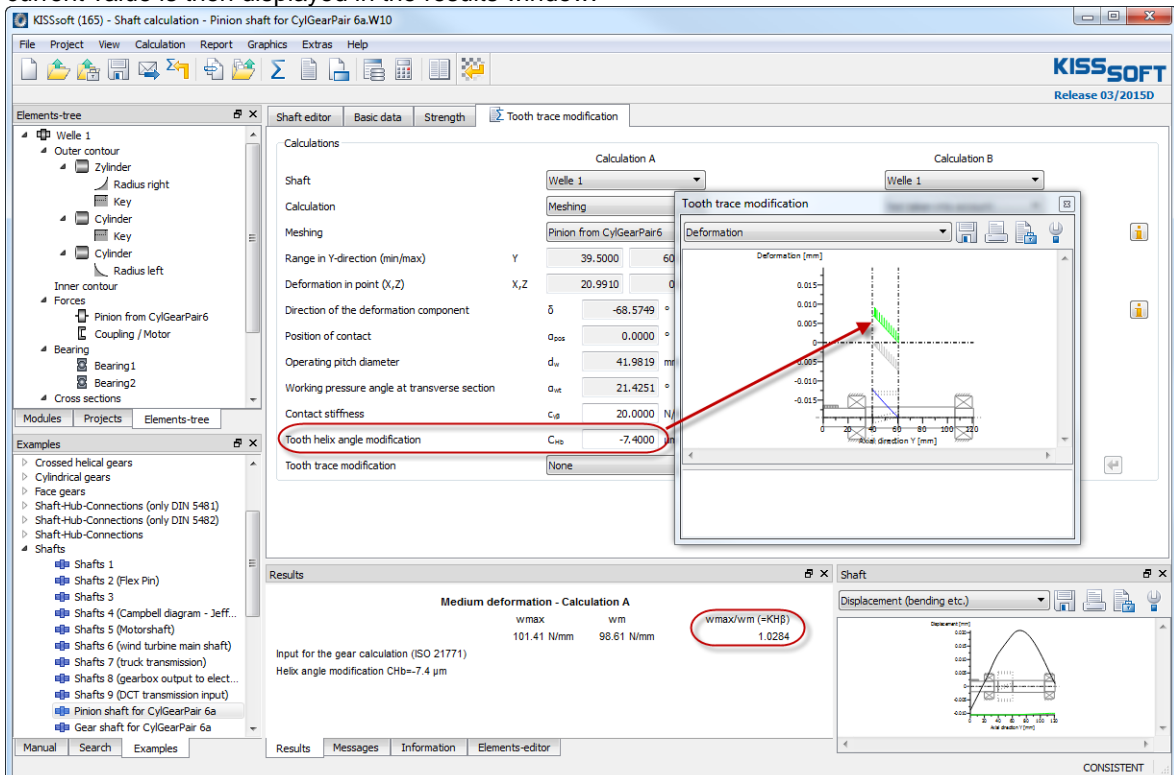



Figure 16. Results overview with enhanced tooth trace modification for the pinion

Since 2012<sup>th</sup> version, the sizing button  also available for this calculation. It runs an internal algorithm to get the optimum result. In the next image, it not possible to see the gray modification anymore as the green defined is now above it. Notice that the value is very close to the one reached before when trying step by step but there is also a small crowning to get is exact. Also the  $K_{H\beta}$  is reduced to 1.0002.

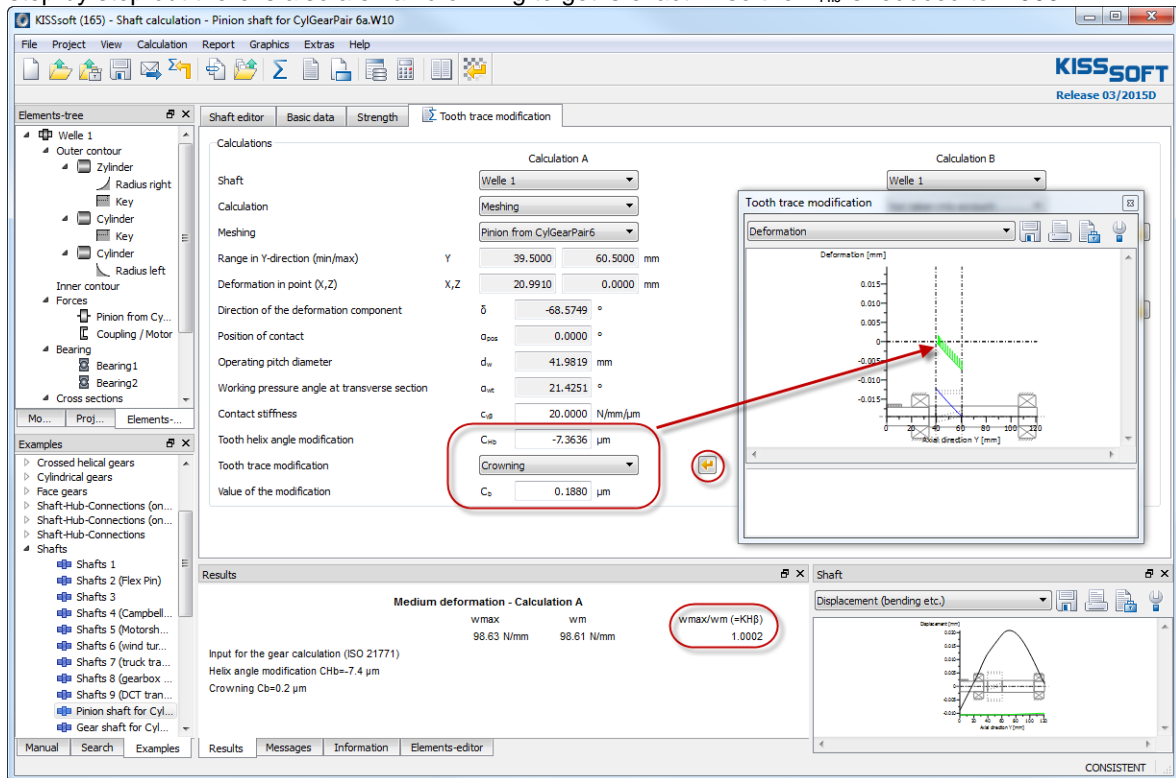


Figure 17. Results overview with optimum tooth trace modification for the pinion

The same should be done for the Gear. Load the file 'Pinion shaft for CylGearPair 6a.W10' and follow the same steps as described for the pinion.

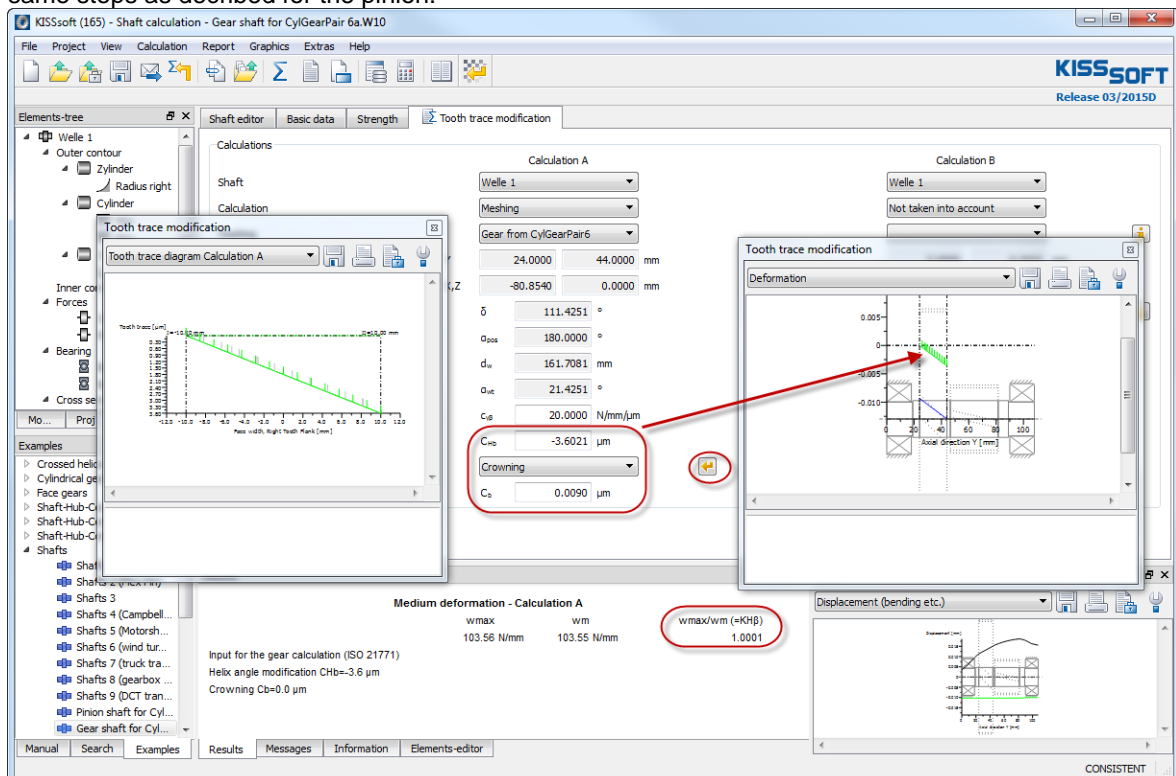


Figure 18. Results overview with optimum tooth trace modification for the gear

## 2.1.4 Step 4: Including modifications in the cylindrical gear calculation

The Shaft Analysis report and the "Deformation" figure both state the tooth flank for which the modification is to be performed. This is calculated from the direction of rotation and the direction of load. In this example, the modification is performed for the right tooth flank for both the pinion and the gear.

Explanations:

...

f.corr : Proposal for modification : **Right tooth flank**

...

Input data: Tooth trace crowned  
 $C_H\beta = -7.4 \mu\text{m}$ ,  $c\beta = 0.2 \mu\text{m}$ ,  $r_{\text{Crown}} = 293287.1 \text{ mm}$

Notice:

The angle modification  $fH_b$  corresponds to a helix angle modification  $-0.0189^\circ$   
 (Helix angle  $14.0591^\circ$ )  
 !! Use modification by helix angle modification only, if the direction of the rotation stays the same!

Figure 19. Report note with optimum tooth trace correction for the pinion

Explanations:

...

f.corr : Proposal for modification : **Right tooth flank**

...

Input data: Tooth trace crowned  
 $C_H\beta = -3.6 \mu\text{m}$ ,  $c\beta = 0.0 \mu\text{m}$ ,  $r_{\text{Crown}} = 5581389.8 \text{ mm}$

Notice:

The angle modification  $fH_b$  corresponds to a helix angle modification  $0.0097^\circ$   
 (Helix angle  $14.0877^\circ$ )  
 !! Use modification by helix angle modification only, if the direction of the rotation stays the same!

Figure 20. Report note with optimum tooth trace modification for the gear

Now open the "Modifications" tab in the cylindrical gear calculation and enter the corrections. You should reduce the helix angle on the pinion and increase the helix angle on the gear.

The screenshot shows the 'Modifications' tab in a software interface. It is divided into sections for Gear 1 and Gear 2. For each gear, there are dropdown menus for 'Start of modification at tip' (set to 'Tip circle') and 'Start of modification at root' (set to 'maximum root form diameter  $d_{fz}$ '). The 'Type of tip modification' is set to 'none'. There are input fields for 'Tip modification' in mm and degrees, and 'Chamfer / tooth end' with values  $b_{\alpha}$  and  $\delta_{2k}$  in mm and degrees. Below this is a table of modifications:

Gear	Flank	Type of modification	Value [ $\mu\text{m}$ ]	Factor 1	Factor 2	Status	Information
Gear 1	both	Tip relief, arc-like	5.0000	0.8651		active	$d_{Ca} = 44.408 \text{ mm}$
Gear 1	both	Crowning	0.2000			active	$r_{\text{crown}} = 275625 \text{ mm}$
Gear 1	both	Helix angle modification, tapered or conical	-7.4000			active	$CH_b = -7.4000 \rightarrow$ Right Tooth Flank beta.ef...
Gear 2	both	Tip relief, arc-like	5.0000	0.5000		active	$d_{Ca} = 163.388 \text{ mm}$
Gear 2	both	Helix angle modification, tapered or conical	-3.6000			active	$CH_b = -3.6000 \rightarrow$ Right Tooth Flank beta.ef...

Figure 21. Inputting a conical helix angle and crowning for the gear and pinion in the Cylindrical gear calculation

The tip reliefs entered in Figure 21 are a good idea, but optional. They do not affect the result. The crowning calculated before by the sizing button is too small to have a real influence, but the optimal value is reached just if applying this modification. It was used also in the Cylindrical gear calculation, shown also in Figure 21 to show how the process works.

## 2.1.5 Step 5: Analyze the optimized situation taking into account the shaft deformation

The calculation of  $K_{Hb}$  in the "Contact analysis" tab now returns a much lower value than previously, due to the flank line modification that has just been applied. Both shaft deformations (pinion and gear) are now included here.

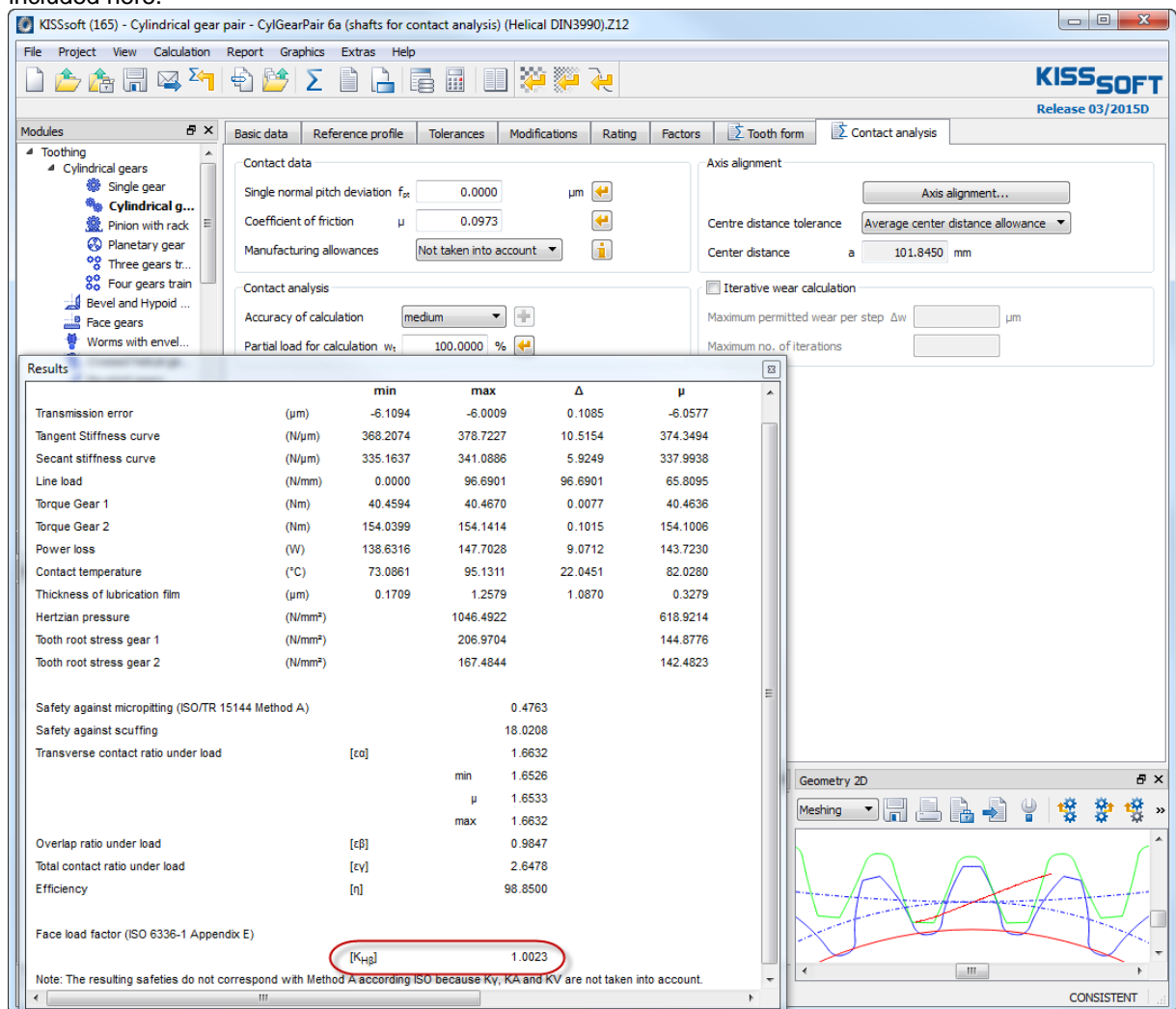


Figure 22. Results overview with optimum tooth trace modification

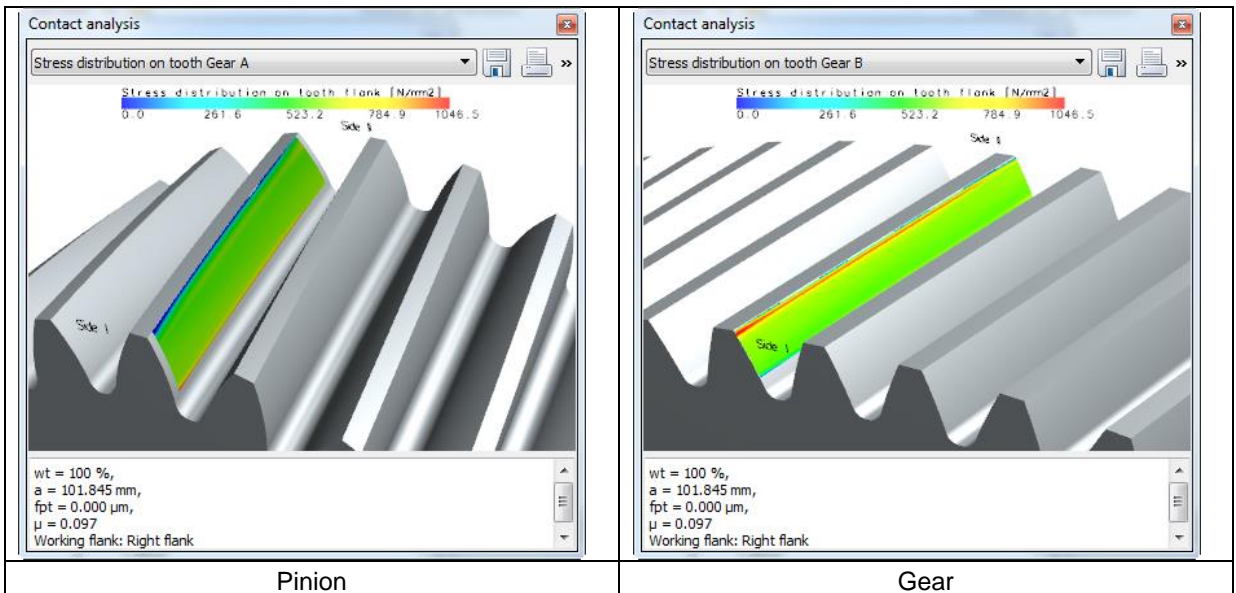


Figure 23. stress distribution calculated with  $K_{H\beta} = 1.0023$

**Note:**

Since 2014<sup>th</sup> version, the sizing function is also available for the gear pair calculation

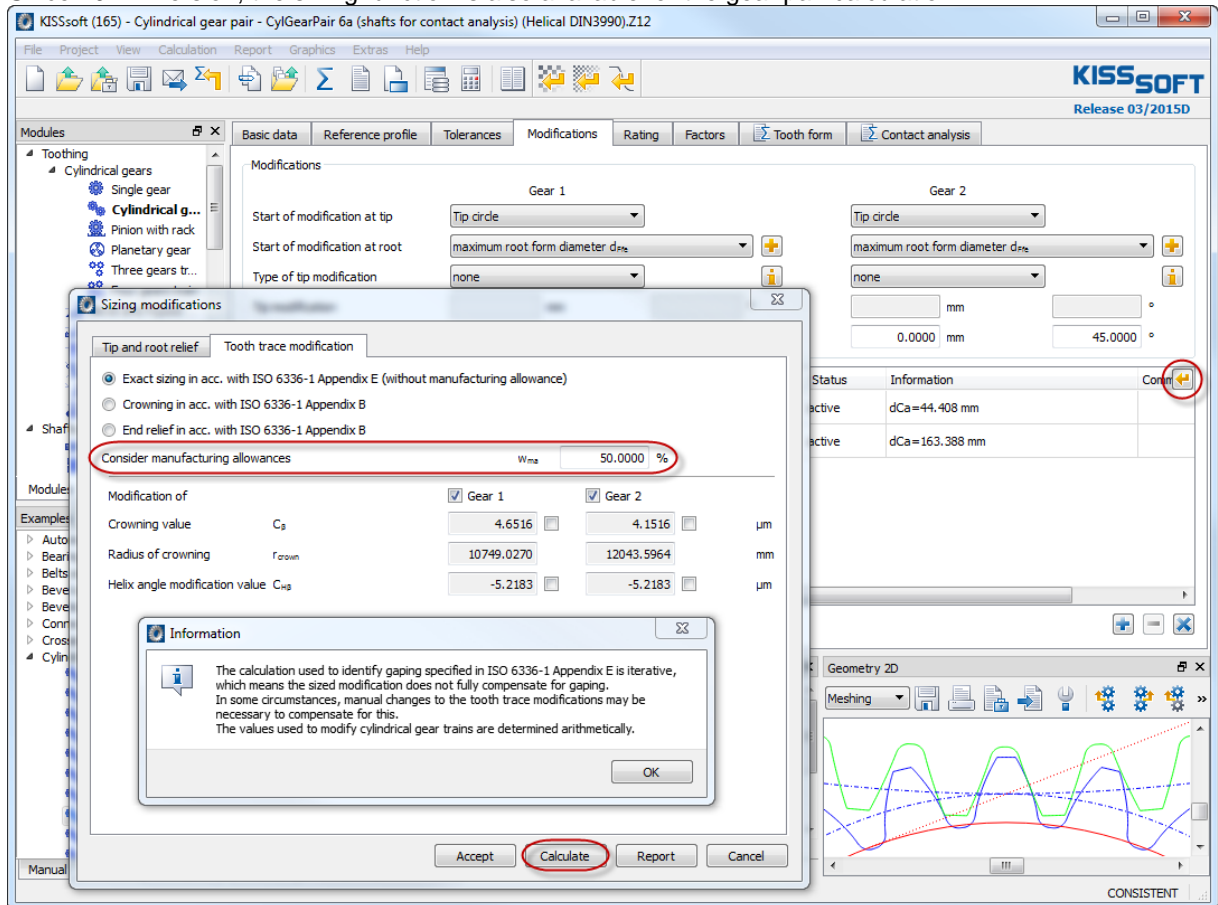


Figure 24. Sizing the tooth trace modification.

## 2.2 Summary

The table below contains an overview of the results.

Step	Face load factors on the	Pinion	Gear	Total	Note
1	$K_{HB}$	2.203	(1.332)		determined in accordance with the standard
2	$K_{HB}$			1.81	taking into account the shafts
3	$K_{HB}$	1.678	1.35		without correction, calculated in the shaft analysis
4	$K_{HB}$	1.029	1.019		with correction, calculated in the shaft analysis
4b	$K_{HB}$	1.0002	1.0001		With optimum correction, calculated in the shaft analysis by sizing button
5	$K_{HB}$			1.0023	with corrections, calculated in the cylindrical gear calculation, and taking into account both shafts



# 3 Results

## 3.1 Report of the contact analysis of the initial variant with main load stage

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KISSsoft Release 03/2015 D

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KISSsoft-Entwicklungs-Version	KISSsoft AG	CH-8608 BUBIKON
File		
Filename: C:/KISSsoft 03-2015/example/CylGearPair 6a (shafts for contact analysis) (Helical DIN3990).Z12		
Description: KISSsoft example		
Changed by: mhoffmann	am: 31.07.2015	um: 13:44:25

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### Contact Analysis

Meshing gear 1 - gear 2

Accuracy of calculation Medium  
**Partial load for calculation**  $[w_t]$  **100.0000 (%)**

Note: In order to obtain contact analysis results for scuffing, micropitting and tooth flank fracture according to method A of ISO the calculation would have to be performed with  $w_t = 100 * K_{gam} * K_A * K_V$ .  
 $(w_t * K_{gam} * K_A * K_V = 107.52 \%)$

Working flank		Right tooth flank
Center distance	[a]	101.8450 (mm)
Single pitch deviation	[f <sub>pt</sub> ]	0.0000 (µm)
Coefficient. of friction	[µ]	0.1073
Deviation error of axis	[f <sub>Σβ</sub> ]	0.0000 (µm)
Inclination error of axis	[f <sub>Σδ</sub> ]	0.0000 (µm)
Torque	[T <sub>1</sub> ]	40.4631 (Nm)

Torsion from shaft calculation

		min	max	Δ	µ	σ
Transmission error	(µm)	-11.0588	-10.1505	0.9083	-10.6135	0.3069
Tangent Stiffness curve	(N/µm)	328.9452	386.7723	57.8271	364.9457	14.0022
Secant stiffness curve	(N/µm)	185.1894	201.1583	15.9689	192.7912	5.4121
Line load	(N/mm)	0.0000	179.4948	179.4948	67.3408	43.7285
Torque Gear 1	(Nm)	40.4592	40.4667	0.0076	40.4624	0.0016
Torque Gear 2	(Nm)	153.6476	154.2387	0.5910	153.8845	0.1903
Power loss	(W)	142.4653	183.2340	40.7687	163.3199	12.8687
Contact temperature	(°C)	73.3086	110.7638	37.4552	82.3447	7.2575
Thickness of lubrication film	(µm)	0.1408	0.9510	0.8102	0.3268	0.1471
Hertzian pressure	(N/mm <sup>2</sup> )		1434.2894		581.3539	
Tooth root stress gear 1	(N/mm <sup>2</sup> )		337.0271		141.9271	
Tooth root stress gear 2	(N/mm <sup>2</sup> )		263.4695		146.1006	

Safety against micropitting (ISO/TR 15144 Method A)		0.3924
Safety against scuffing		11.1099
Transverse contact ratio under load	[εα]	1.7684
	min	1.3579
	µ	1.6177
	max	1.7684
Overlap ratio under load	[εβ]	0.7426
Total contact ratio under load	[εγ]	2.5110
Efficiency	[η]	98.6900

Face load factor (ISO 6336-1 Appendix E)

Note: The resulting safeties do not correspond with Method A according ISO because K<sub>γ</sub>, K<sub>A</sub> and K<sub>V</sub> are not taken into account.

Amplitude spectrum of the transmission error

Harmonics	Amplitude (μm)
1	0.427
2	0.057
3	0.013
4	0.006
5	0.005
6	0.003
7	0.001
8	0.001
9	0.001
10	0.001

### K<sub>Hβ</sub> Calculation - Gear 1 - Gear 2

Right flank

Shaft file A: C:/KISSsoft 03-2015/example/Pinion shaft for CylGearPair 6a.W10, selected gear: Pinion from CylGearPair6 (gears as masses and stiffness)

Shaft file B: C:/KISSsoft 03-2015/example/Gear shaft for CylGearPair 6a.W10, selected gear: Gear from CylGearPair6 (gears as masses and stiffness)

(Partial load for calculation  $w_t = 100\%$ )

$f_{ma} = 0.000 \mu\text{m}$ ,  $f_{H\beta} = 0.000 \mu\text{m}$

Result after  $i = 11$  iterations of load distribution

Gear 1

Point in polar co-ordinates:

$R = 20.991 \text{ mm}$ ,  $\varphi = 0.000^\circ$

Displacement calculated in direction  $111.425^\circ$

	y	$\varphi_{1.t}$	f1.t	f1.b	f1.tot	f1.C	f1.tot+f1.C
1	40.196 mm	0.0000°	-0.0045 μm	-0.0390 μm	-0.0435 μm	0.0000 μm	-0.0435 μm
2	40.588 mm	-0.0000°	0.0094 μm	0.0820 μm	0.0914 μm	0.0000 μm	0.0914 μm
3	40.980 mm	-0.0001°	0.0232 μm	0.2030 μm	0.2262 μm	0.0000 μm	0.2262 μm
4	41.373 mm	-0.0001°	0.0364 μm	0.3234 μm	0.3599 μm	0.0000 μm	0.3599 μm
5	41.765 mm	-0.0001°	0.0494 μm	0.4436 μm	0.4930 μm	0.0000 μm	0.4930 μm
6	42.157 mm	-0.0002°	0.0619 μm	0.5634 μm	0.6253 μm	0.0000 μm	0.6253 μm
7	42.549 mm	-0.0002°	0.0737 μm	0.6828 μm	0.7565 μm	0.0000 μm	0.7565 μm
8	42.941 mm	-0.0003°	0.0853 μm	0.8020 μm	0.8872 μm	0.0000 μm	0.8872 μm
9	43.333 mm	-0.0003°	0.0961 μm	0.9206 μm	1.0167 μm	0.0000 μm	1.0167 μm
10	43.725 mm	-0.0003°	0.1067 μm	1.0389 μm	1.1456 μm	0.0000 μm	1.1456 μm
11	44.118 mm	-0.0003°	0.1168 μm	1.1570 μm	1.2738 μm	0.0000 μm	1.2738 μm
12	44.510 mm	-0.0004°	0.1264 μm	1.2745 μm	1.4009 μm	0.0000 μm	1.4009 μm
13	44.902 mm	-0.0004°	0.1357 μm	1.3919 μm	1.5276 μm	0.0000 μm	1.5276 μm
14	45.294 mm	-0.0004°	0.1444 μm	1.5088 μm	1.6532 μm	0.0000 μm	1.6532 μm
15	45.686 mm	-0.0004°	0.1528 μm	1.6254 μm	1.7782 μm	0.0000 μm	1.7782 μm
16	46.078 mm	-0.0005°	0.1609 μm	1.7417 μm	1.9026 μm	0.0000 μm	1.9026 μm
17	46.471 mm	-0.0005°	0.1684 μm	1.8576 μm	2.0260 μm	0.0000 μm	2.0260 μm
18	46.863 mm	-0.0005°	0.1757 μm	1.9732 μm	2.1490 μm	0.0000 μm	2.1490 μm
19	47.255 mm	-0.0005°	0.1826 μm	2.0885 μm	2.2710 μm	0.0000 μm	2.2710 μm
20	47.647 mm	-0.0006°	0.1890 μm	2.2034 μm	2.3925 μm	0.0000 μm	2.3925 μm
21	48.039 mm	-0.0006°	0.1953 μm	2.3182 μm	2.5135 μm	0.0000 μm	2.5135 μm
22	48.431 mm	-0.0006°	0.2010 μm	2.4324 μm	2.6334 μm	0.0000 μm	2.6334 μm
23	48.824 mm	-0.0006°	0.2066 μm	2.5465 μm	2.7530 μm	0.0000 μm	2.7530 μm
24	49.216 mm	-0.0006°	0.2117 μm	2.6602 μm	2.8719 μm	0.0000 μm	2.8719 μm

25	49.608 mm	-0.0006°	0.2165 µm	2.7736 µm	2.9901 µm	0.0000 µm	2.9901 µm
26	50.000 mm	-0.0006°	0.2212 µm	2.8868 µm	3.1080 µm	0.0000 µm	3.1080 µm
27	50.392 mm	-0.0007°	0.2253 µm	2.9996 µm	3.2249 µm	0.0000 µm	3.2249 µm
28	50.784 mm	-0.0007°	0.2293 µm	3.1122 µm	3.3415 µm	0.0000 µm	3.3415 µm
29	51.176 mm	-0.0007°	0.2331 µm	3.2244 µm	3.4575 µm	0.0000 µm	3.4575 µm
30	51.569 mm	-0.0007°	0.2365 µm	3.3364 µm	3.5729 µm	0.0000 µm	3.5729 µm
31	51.961 mm	-0.0007°	0.2397 µm	3.4482 µm	3.6879 µm	0.0000 µm	3.6879 µm
32	52.353 mm	-0.0007°	0.2426 µm	3.5596 µm	3.8022 µm	0.0000 µm	3.8022 µm
33	52.745 mm	-0.0007°	0.2453 µm	3.6708 µm	3.9161 µm	0.0000 µm	3.9161 µm
34	53.137 mm	-0.0007°	0.2478 µm	3.7818 µm	4.0295 µm	0.0000 µm	4.0295 µm
35	53.529 mm	-0.0007°	0.2500 µm	3.8924 µm	4.1424 µm	0.0000 µm	4.1424 µm
36	53.922 mm	-0.0007°	0.2520 µm	4.0029 µm	4.2550 µm	0.0000 µm	4.2550 µm
37	54.314 mm	-0.0007°	0.2538 µm	4.1131 µm	4.3669 µm	0.0000 µm	4.3669 µm
38	54.706 mm	-0.0007°	0.2555 µm	4.2231 µm	4.4785 µm	0.0000 µm	4.4785 µm
39	55.098 mm	-0.0008°	0.2570 µm	4.3328 µm	4.5898 µm	0.0000 µm	4.5898 µm
40	55.490 mm	-0.0008°	0.2582 µm	4.4423 µm	4.7005 µm	0.0000 µm	4.7005 µm
41	55.882 mm	-0.0008°	0.2593 µm	4.5516 µm	4.8110 µm	0.0000 µm	4.8110 µm
42	56.275 mm	-0.0008°	0.2603 µm	4.6607 µm	4.9209 µm	0.0000 µm	4.9209 µm
43	56.667 mm	-0.0008°	0.2611 µm	4.7695 µm	5.0306 µm	0.0000 µm	5.0306 µm
44	57.059 mm	-0.0008°	0.2618 µm	4.8782 µm	5.1400 µm	0.0000 µm	5.1400 µm
45	57.451 mm	-0.0008°	0.2623 µm	4.9866 µm	5.2489 µm	0.0000 µm	5.2489 µm
46	57.843 mm	-0.0008°	0.2627 µm	5.0949 µm	5.3576 µm	0.0000 µm	5.3576 µm
47	58.235 mm	-0.0008°	0.2630 µm	5.2030 µm	5.4659 µm	0.0000 µm	5.4659 µm
48	58.627 mm	-0.0008°	0.2631 µm	5.3108 µm	5.5740 µm	0.0000 µm	5.5740 µm
49	59.020 mm	-0.0008°	0.2633 µm	5.4186 µm	5.6818 µm	0.0000 µm	5.6818 µm
50	59.412 mm	-0.0008°	0.2633 µm	5.5260 µm	5.7893 µm	0.0000 µm	5.7893 µm
51	59.804 mm	-0.0008°	0.2633 µm	5.6335 µm	5.8968 µm	0.0000 µm	5.8968 µm

#### Gear 2

Point in polar co-ordinates:

R = 80.854 mm ,  $\varphi = 180.000^\circ$

Displacement calculated in direction 111.425 °

	y	$\varphi_{2.t}$	f2.t	f2.b	f2.tot	f2.C	f2.tot+f2.C
1	24.196 mm	-0.0000°	-0.0000 µm	0.0226 µm	0.0226 µm	0.0000 µm	0.0226 µm
2	24.588 mm	0.0000°	0.0000 µm	-0.0475 µm	-0.0475 µm	-0.0000 µm	-0.0475 µm
3	24.980 mm	0.0000°	0.0000 µm	-0.1176 µm	-0.1176 µm	-0.0000 µm	-0.1176 µm
4	25.373 mm	0.0000°	0.0001 µm	-0.1877 µm	-0.1876 µm	-0.0000 µm	-0.1876 µm
5	25.765 mm	0.0000°	0.0002 µm	-0.2578 µm	-0.2576 µm	-0.0000 µm	-0.2576 µm
6	26.157 mm	0.0000°	0.0003 µm	-0.3279 µm	-0.3275 µm	-0.0000 µm	-0.3275 µm
7	26.549 mm	0.0000°	0.0005 µm	-0.3979 µm	-0.3974 µm	-0.0000 µm	-0.3974 µm
8	26.941 mm	0.0000°	0.0007 µm	-0.4679 µm	-0.4672 µm	-0.0000 µm	-0.4672 µm
9	27.333 mm	0.0000°	0.0009 µm	-0.5379 µm	-0.5370 µm	-0.0000 µm	-0.5370 µm
10	27.725 mm	0.0000°	0.0011 µm	-0.6079 µm	-0.6068 µm	-0.0000 µm	-0.6068 µm
11	28.118 mm	0.0000°	0.0014 µm	-0.6779 µm	-0.6765 µm	-0.0000 µm	-0.6765 µm
12	28.510 mm	0.0000°	0.0017 µm	-0.7478 µm	-0.7461 µm	-0.0000 µm	-0.7461 µm
13	28.902 mm	0.0000°	0.0020 µm	-0.8178 µm	-0.8158 µm	-0.0000 µm	-0.8158 µm
14	29.294 mm	0.0000°	0.0024 µm	-0.8877 µm	-0.8853 µm	-0.0000 µm	-0.8853 µm
15	29.686 mm	0.0000°	0.0027 µm	-0.9576 µm	-0.9549 µm	-0.0000 µm	-0.9549 µm
16	30.078 mm	0.0000°	0.0031 µm	-1.0275 µm	-1.0244 µm	-0.0000 µm	-1.0244 µm
17	30.471 mm	0.0000°	0.0035 µm	-1.0974 µm	-1.0938 µm	-0.0000 µm	-1.0938 µm
18	30.863 mm	0.0000°	0.0040 µm	-1.1672 µm	-1.1633 µm	-0.0000 µm	-1.1633 µm
19	31.255 mm	0.0000°	0.0044 µm	-1.2371 µm	-1.2326 µm	-0.0000 µm	-1.2326 µm
20	31.647 mm	0.0000°	0.0049 µm	-1.3069 µm	-1.3020 µm	-0.0000 µm	-1.3020 µm
21	32.039 mm	0.0000°	0.0054 µm	-1.3767 µm	-1.3713 µm	-0.0000 µm	-1.3713 µm
22	32.431 mm	0.0000°	0.0060 µm	-1.4465 µm	-1.4406 µm	-0.0000 µm	-1.4406 µm
23	32.824 mm	0.0000°	0.0065 µm	-1.5163 µm	-1.5098 µm	-0.0000 µm	-1.5098 µm
24	33.216 mm	0.0000°	0.0071 µm	-1.5861 µm	-1.5790 µm	-0.0000 µm	-1.5790 µm

25	33.608 mm	0.0000°	0.0077 μm	-1.6558 μm	-1.6482 μm	-0.0000 μm	-1.6482 μm
26	34.000 mm	0.0000°	0.0083 μm	-1.7256 μm	-1.7173 μm	-0.0000 μm	-1.7173 μm
27	34.392 mm	0.0000°	0.0089 μm	-1.7953 μm	-1.7865 μm	-0.0000 μm	-1.7865 μm
28	34.784 mm	0.0000°	0.0095 μm	-1.8651 μm	-1.8555 μm	-0.0000 μm	-1.8555 μm
29	35.176 mm	0.0000°	0.0102 μm	-1.9348 μm	-1.9246 μm	-0.0000 μm	-1.9246 μm
30	35.569 mm	0.0000°	0.0108 μm	-2.0045 μm	-1.9936 μm	-0.0000 μm	-1.9936 μm
31	35.961 mm	0.0000°	0.0115 μm	-2.0742 μm	-2.0626 μm	-0.0000 μm	-2.0626 μm
32	36.353 mm	0.0000°	0.0122 μm	-2.1438 μm	-2.1316 μm	-0.0000 μm	-2.1316 μm
33	36.745 mm	0.0000°	0.0129 μm	-2.2135 μm	-2.2006 μm	-0.0000 μm	-2.2006 μm
34	37.137 mm	0.0000°	0.0137 μm	-2.2832 μm	-2.2695 μm	-0.0000 μm	-2.2695 μm
35	37.529 mm	0.0000°	0.0144 μm	-2.3528 μm	-2.3384 μm	-0.0000 μm	-2.3384 μm
36	37.922 mm	0.0000°	0.0152 μm	-2.4224 μm	-2.4073 μm	-0.0000 μm	-2.4073 μm
37	38.314 mm	0.0000°	0.0159 μm	-2.4921 μm	-2.4761 μm	-0.0000 μm	-2.4761 μm
38	38.706 mm	0.0000°	0.0167 μm	-2.5617 μm	-2.5449 μm	-0.0000 μm	-2.5449 μm
39	39.098 mm	0.0000°	0.0175 μm	-2.6313 μm	-2.6138 μm	-0.0000 μm	-2.6138 μm
40	39.490 mm	0.0000°	0.0183 μm	-2.7009 μm	-2.6825 μm	-0.0000 μm	-2.6825 μm
41	39.882 mm	0.0000°	0.0191 μm	-2.7705 μm	-2.7513 μm	-0.0000 μm	-2.7513 μm
42	40.275 mm	0.0000°	0.0200 μm	-2.8400 μm	-2.8201 μm	-0.0000 μm	-2.8201 μm
43	40.667 mm	0.0000°	0.0208 μm	-2.9096 μm	-2.8888 μm	-0.0000 μm	-2.8888 μm
44	41.059 mm	0.0000°	0.0216 μm	-2.9792 μm	-2.9575 μm	-0.0000 μm	-2.9575 μm
45	41.451 mm	0.0000°	0.0225 μm	-3.0487 μm	-3.0262 μm	-0.0000 μm	-3.0262 μm
46	41.843 mm	0.0000°	0.0234 μm	-3.1183 μm	-3.0949 μm	-0.0000 μm	-3.0949 μm
47	42.235 mm	0.0000°	0.0242 μm	-3.1878 μm	-3.1636 μm	-0.0000 μm	-3.1636 μm
48	42.627 mm	0.0000°	0.0251 μm	-3.2574 μm	-3.2323 μm	-0.0000 μm	-3.2323 μm
49	43.020 mm	0.0000°	0.0260 μm	-3.3269 μm	-3.3009 μm	-0.0000 μm	-3.3009 μm
50	43.412 mm	0.0000°	0.0269 μm	-3.3964 μm	-3.3696 μm	-0.0000 μm	-3.3696 μm
51	43.804 mm	0.0000°	0.0278 μm	-3.4659 μm	-3.4382 μm	-0.0000 μm	-3.4382 μm

Explanations:

y	: Width
φ.t	: Static torsion
f.t	: Displacement due to torsion
f.b	: Displacement due to bending
f.tot	: Total displacement (f.b+f.t)
f.C	: Change due to flank line modification

Load distribution

Contact stiffness = 18.895 N/mm/μm

Young's modulus = 206000.0/206000.0 N/mm<sup>2</sup>

	y	δ	g	w
1.	40.1961 mm	0.0000 μm	10.7505 μm	203.1328 N/mm
2.	40.5882 mm	0.2050 μm	10.5455 μm	199.2601 N/mm
3.	40.9804 mm	0.4099 μm	10.3406 μm	195.3882 N/mm
4.	41.3725 mm	0.6136 μm	10.1369 μm	191.5390 N/mm
5.	41.7647 mm	0.8167 μm	9.9338 μm	187.7008 N/mm
6.	42.1569 mm	1.0189 μm	9.7315 μm	183.8802 N/mm
7.	42.5490 mm	1.2200 μm	9.5305 μm	180.0804 N/mm
8.	42.9412 mm	1.4206 μm	9.3299 μm	176.2906 N/mm
9.	43.3333 mm	1.6199 μm	9.1306 μm	172.5253 N/mm
10.	43.7255 mm	1.8185 μm	8.9320 μm	168.7718 N/mm
11.	44.1176 mm	2.0164 μm	8.7341 μm	165.0328 N/mm
12.	44.5098 mm	2.2132 μm	8.5373 μm	161.3141 N/mm
13.	44.9020 mm	2.4095 μm	8.3410 μm	157.6045 N/mm
14.	45.2941 mm	2.6047 μm	8.1458 μm	153.9170 N/mm
15.	45.6863 mm	2.7992 μm	7.9513 μm	150.2414 N/mm
16.	46.0784 mm	2.9931 μm	7.7573 μm	146.5770 N/mm
17.	46.4706 mm	3.1860 μm	7.5645 μm	142.9333 N/mm
18.	46.8627 mm	3.3784 μm	7.3721 μm	139.2979 N/mm

19.	47.2549 mm	3.5698 $\mu\text{m}$	7.1807 $\mu\text{m}$	135.6805 N/mm
20.	47.6471 mm	3.7606 $\mu\text{m}$	6.9899 $\mu\text{m}$	132.0758 N/mm
21.	48.0392 mm	3.9509 $\mu\text{m}$	6.7996 $\mu\text{m}$	128.4802 N/mm
22.	48.4314 mm	4.1401 $\mu\text{m}$	6.6104 $\mu\text{m}$	124.9044 N/mm
23.	48.8235 mm	4.3290 $\mu\text{m}$	6.4215 $\mu\text{m}$	121.3359 N/mm
24.	49.2157 mm	4.5171 $\mu\text{m}$	6.2334 $\mu\text{m}$	117.7819 N/mm
25.	49.6078 mm	4.7044 $\mu\text{m}$	6.0460 $\mu\text{m}$	114.2412 N/mm
26.	50.0000 mm	4.8914 $\mu\text{m}$	5.8590 $\mu\text{m}$	110.7078 N/mm
27.	50.3922 mm	5.0775 $\mu\text{m}$	5.6730 $\mu\text{m}$	107.1924 N/mm
28.	50.7843 mm	5.2632 $\mu\text{m}$	5.4873 $\mu\text{m}$	103.6838 N/mm
29.	51.1765 mm	5.4482 $\mu\text{m}$	5.3023 $\mu\text{m}$	100.1875 N/mm
30.	51.5686 mm	5.6326 $\mu\text{m}$	5.1179 $\mu\text{m}$	96.7036 N/mm
31.	51.9608 mm	5.8166 $\mu\text{m}$	4.9338 $\mu\text{m}$	93.2258 N/mm
32.	52.3529 mm	5.9999 $\mu\text{m}$	4.7506 $\mu\text{m}$	89.7636 N/mm
33.	52.7451 mm	6.1828 $\mu\text{m}$	4.5677 $\mu\text{m}$	86.3081 N/mm
34.	53.1373 mm	6.3651 $\mu\text{m}$	4.3853 $\mu\text{m}$	82.8618 N/mm
35.	53.5294 mm	6.5469 $\mu\text{m}$	4.2036 $\mu\text{m}$	79.4276 N/mm
36.	53.9216 mm	6.7284 $\mu\text{m}$	4.0221 $\mu\text{m}$	75.9988 N/mm
37.	54.3137 mm	6.9091 $\mu\text{m}$	3.8413 $\mu\text{m}$	72.5828 N/mm
38.	54.7059 mm	7.0896 $\mu\text{m}$	3.6609 $\mu\text{m}$	69.1732 N/mm
39.	55.0980 mm	7.2697 $\mu\text{m}$	3.4808 $\mu\text{m}$	65.7703 N/mm
40.	55.4902 mm	7.4492 $\mu\text{m}$	3.3013 $\mu\text{m}$	62.3791 N/mm
41.	55.8824 mm	7.6284 $\mu\text{m}$	3.1221 $\mu\text{m}$	58.9924 N/mm
42.	56.2745 mm	7.8071 $\mu\text{m}$	2.9433 $\mu\text{m}$	55.6153 N/mm
43.	56.6667 mm	7.9855 $\mu\text{m}$	2.7650 $\mu\text{m}$	52.2446 N/mm
44.	57.0588 mm	8.1636 $\mu\text{m}$	2.5868 $\mu\text{m}$	48.8791 N/mm
45.	57.4510 mm	8.3412 $\mu\text{m}$	2.4092 $\mu\text{m}$	45.5233 N/mm
46.	57.8431 mm	8.5186 $\mu\text{m}$	2.2318 $\mu\text{m}$	42.1712 N/mm
47.	58.2353 mm	8.6956 $\mu\text{m}$	2.0548 $\mu\text{m}$	38.8265 N/mm
48.	58.6275 mm	8.8723 $\mu\text{m}$	1.8781 $\mu\text{m}$	35.4876 N/mm
49.	59.0196 mm	9.0489 $\mu\text{m}$	1.7016 $\mu\text{m}$	32.1523 N/mm
50.	59.4118 mm	9.2250 $\mu\text{m}$	1.5255 $\mu\text{m}$	28.8242 N/mm
51.	59.8039 mm	9.4011 $\mu\text{m}$	1.3493 $\mu\text{m}$	25.4963 N/mm

Explanations:

$\delta$  : Gap

g : Flank overlap

w : Line load

Force application point, Y direction:  $y = 46.444 \text{ mm}$  ( $F = 2239.3 \text{ N}$ )

To take into account the load distribution in the shaft calculation: Force center point offset:  $\Delta y = -3.556 \text{ mm}$

$w_{\text{max}} = 203.133 \text{ N/mm}$ ,  $w_{\text{m}} = 111.964 \text{ N/mm}$

$w_{\text{m}} = K_{\text{V}} * K_{\text{A}} * K_{\text{Y}} * (F_{\text{t}}/b) / \cos(\alpha_{\text{wt}})$

$K_{\text{V}} = 1.0752$ ,  $K_{\text{A}} = 1.000$ ,  $K_{\text{Y}} = 1.000$

$K_{\text{H}\beta} = w_{\text{max}}/w_{\text{m}} = 1.8143$  (Calculation according to ISO 6336-1, Appendix E)

Side I, II:  $w_{\text{I}} / w_{\text{m}} = 1.8143$   $w_{\text{II}} / w_{\text{m}} = 0.2277$

Notice: The influence of the exceeding facewidth is not taken into account in the calculation of  $K_{\text{H}\beta}$ .

### Calculation of the 'equivalent' linear misalignment of the shafts

a) Calculation of the 'equivalent' linear misalignment, so that  $K_{\text{H}\beta}$  remains the same

Linear misalignment in the plane of action, Gear 1:  $[F_{\beta x1}] 6.18 \mu\text{m}$  ( $K_{\text{H}\beta1} = 1.5211$ )

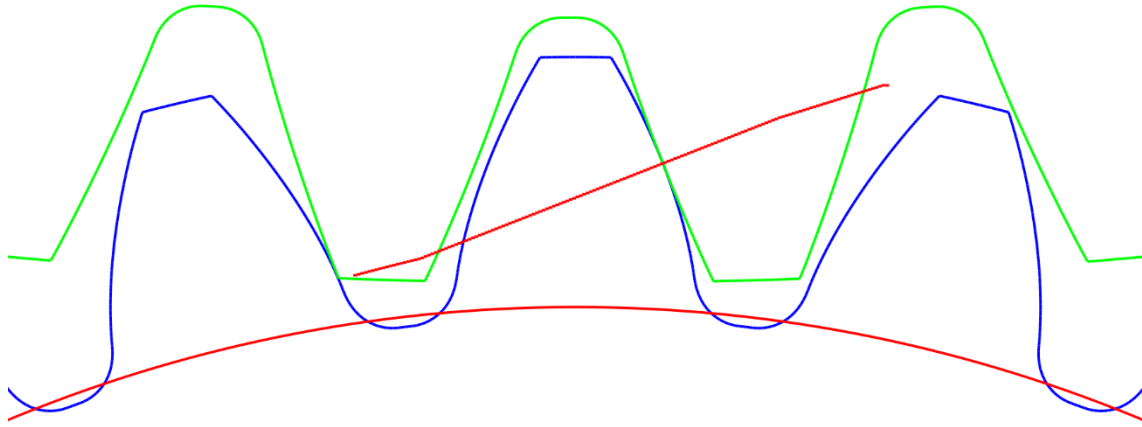
Linear misalignment in the plane of action, Gear 2:  $[F_{\beta x2}] 3.47 \mu\text{m}$  ( $K_{\text{H}\beta2} = 1.2931$ )

Axis alignment: Deviation error of axis/Inclination error of axis  $f_{\Sigma\beta}/f_{\Sigma\delta} = 8.98 \mu\text{m}/3.52 \mu\text{m}$

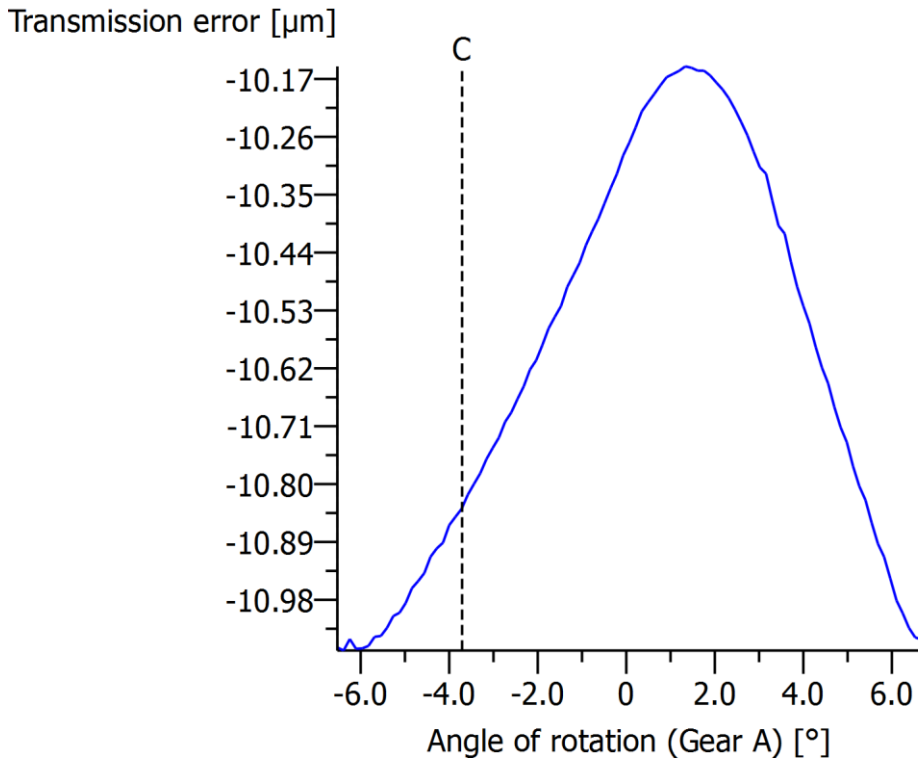
( $F_{\beta x} = 9.65 \mu\text{m}$ )

The deformation due to torsion is included in the specification of  $f_{\Sigma\beta}/f_{\Sigma\delta}$ .

b) Linear misalignment due to connection of the gap from side I to II  
 Linear misalignment in the plane of action, Gear 1:  $[F_{\beta x1}]$  5.94  $\mu\text{m}$   
 Linear misalignment in the plane of action, Gear 2:  $[F_{\beta x2}]$  3.46  $\mu\text{m}$   
 Axis alignment: Deviation error of axis/Inclination error of axis  $f\Sigma\beta/f\Sigma\delta$ : 8.75  $\mu\text{m}/3.43$   $\mu\text{m}$  ( $F_{\beta x} = 9.40$   $\mu\text{m}$ )  
 The deformation due to torsion is included in the specification of  $f\Sigma\beta/f\Sigma\delta$ .

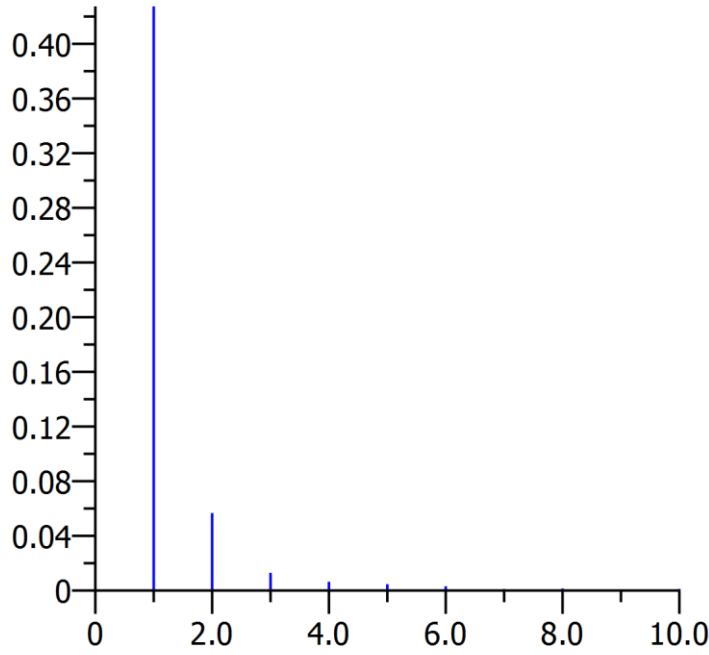


da1 = 45.6874 mm, df1 = 38.8180 mm, As1 = -0.0540 mm, da2 = 163.9684 mm, df2 = 156.9589 mm, As2 = -0.1050 mm  
 Figure: Path of contact



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$  Working flank: Right flank  
 Figure: Transmission error

Amplitude [ $\mu\text{m}$ ]



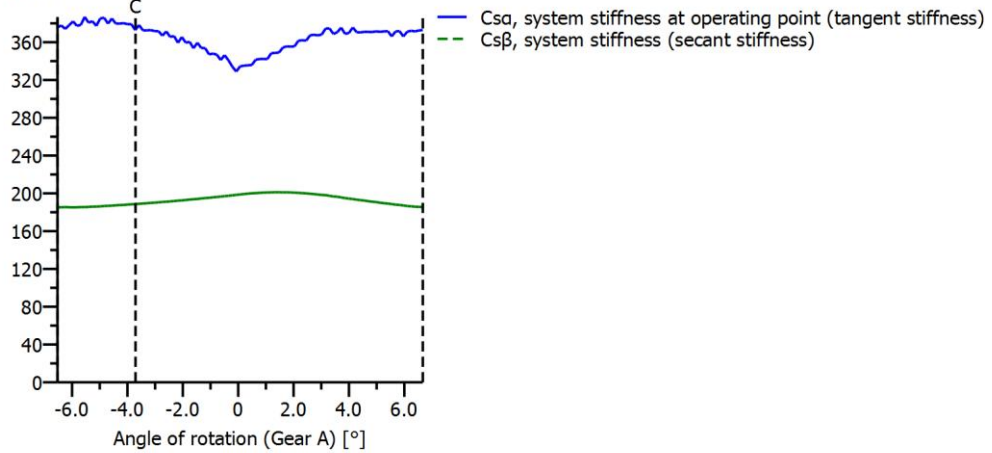
Arrangement of harmonics

Arrangement of harmonics	Amplitude [ $\mu\text{m}$ ]	1.	0.4273782.	0.0566563.	0.0129554.	0.0064075.
0.0046326.	0.0031277.	0.0011528.	0.0014699.	0.00062510.	0.001185	

wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu$  = 0.107 Working flank: Right flank

Figure: Amplitude spectrum of transmission error

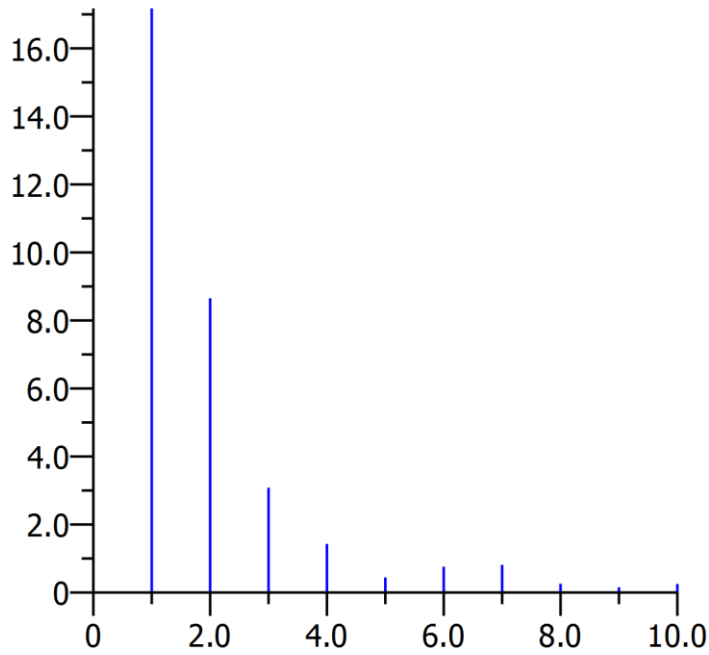
System stiffness [ $\text{N}/\mu\text{m}$ ]



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu$  = 0.107 Working flank: Right flank  $C_{s\alpha\_mean} = 364.8450387 \text{ N}/\mu\text{m}$   $C_{s\beta\_mean} = 192.8693904 \text{ N}/\mu\text{m}$   $C_{s\alpha} = C_{y\alpha} * b$   $C_{s\beta} = C_{y\beta} * b$

Figure: Stiffness curve

Amplitude [ $\mu\text{m}$ ]



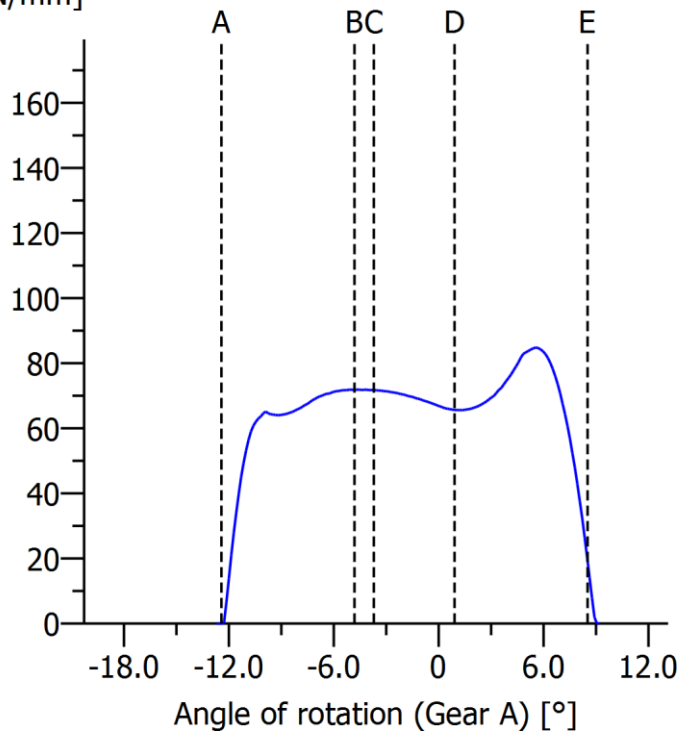
Arrangement of harmonics

Amplitude [N/mm/ $\mu\text{m}$ ]	17.1747232	8.6523893	3.0807524
1.4289105	0.4439176	0.7597007	0.8163658
0.251625	wt = 100 %	a = 101.845 mm	fpt = 0.000 $\mu\text{m}$ , $\mu = 0.107$

Working flank: Right flank

Figure: Amplitude spectrum of contact stiffness

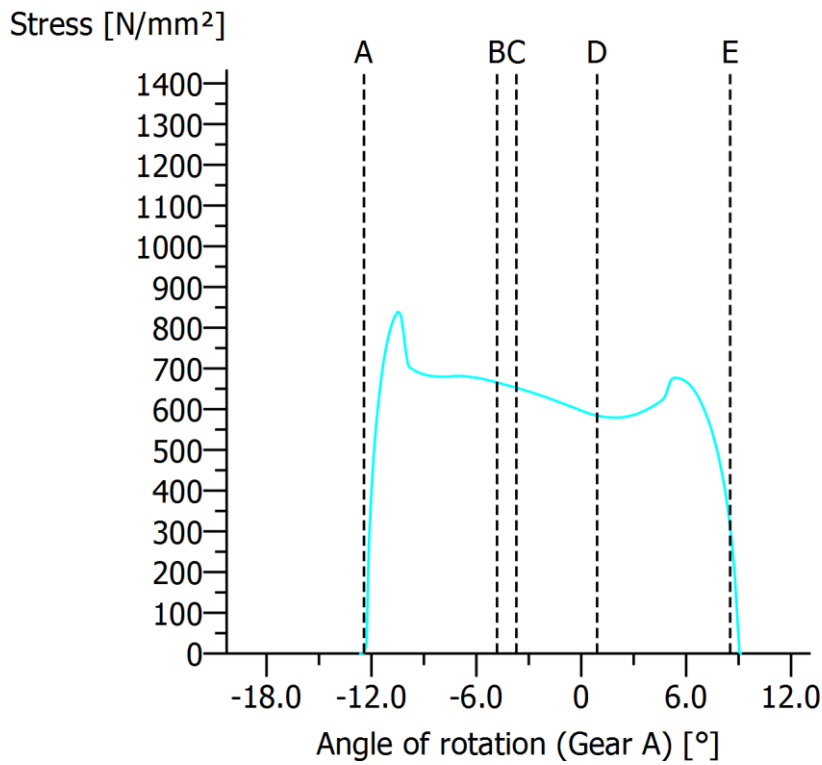
Normal force (line load) [N/mm]



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$  Working flank: Right flank

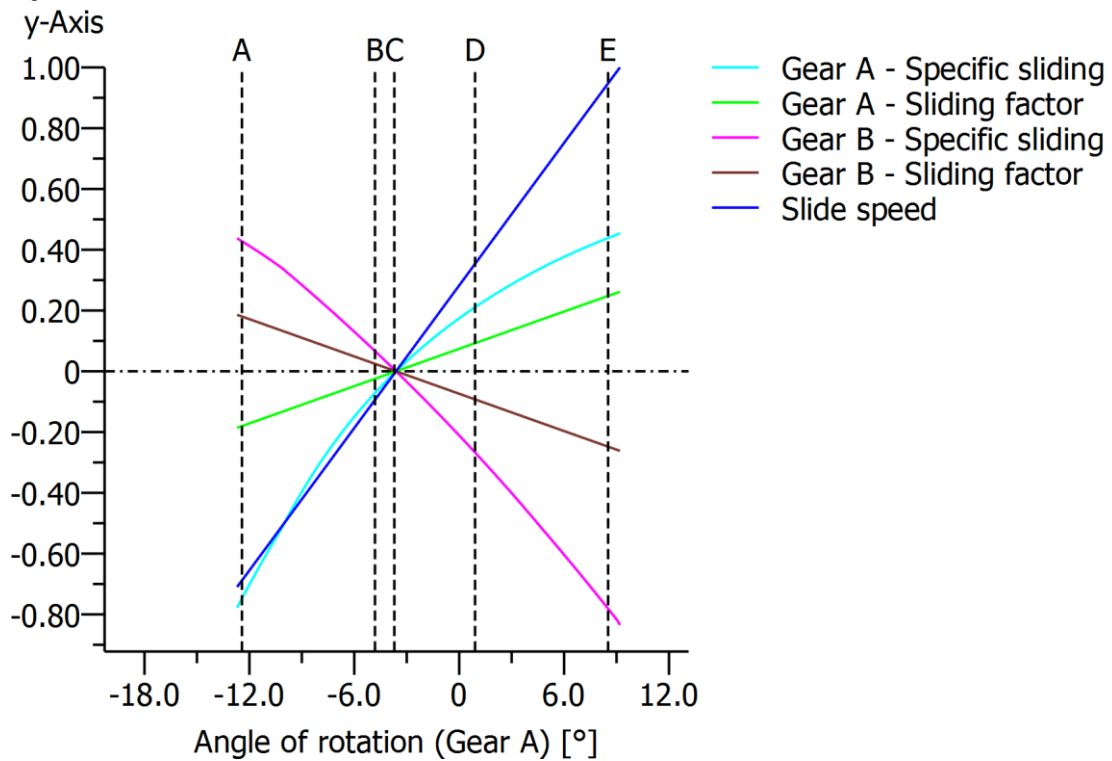
Figure: Normal force curve (Line load)





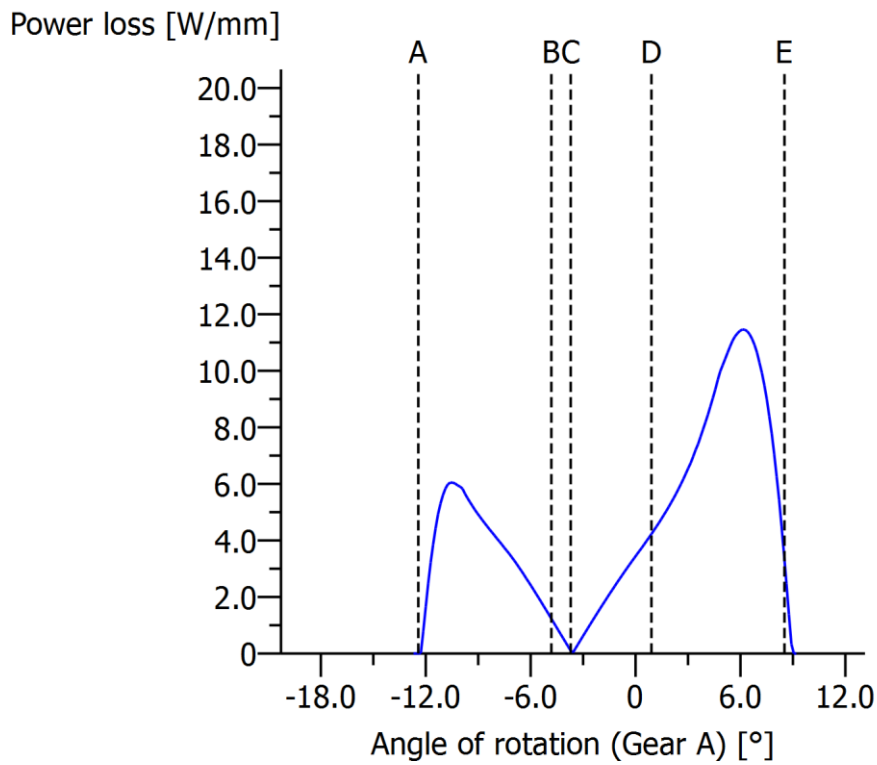
wt = 100 %, a = 101.845 mm, fpt = 0.000 μm, μ = 0.107 Working flank: Right flank

Figure: Stress curve

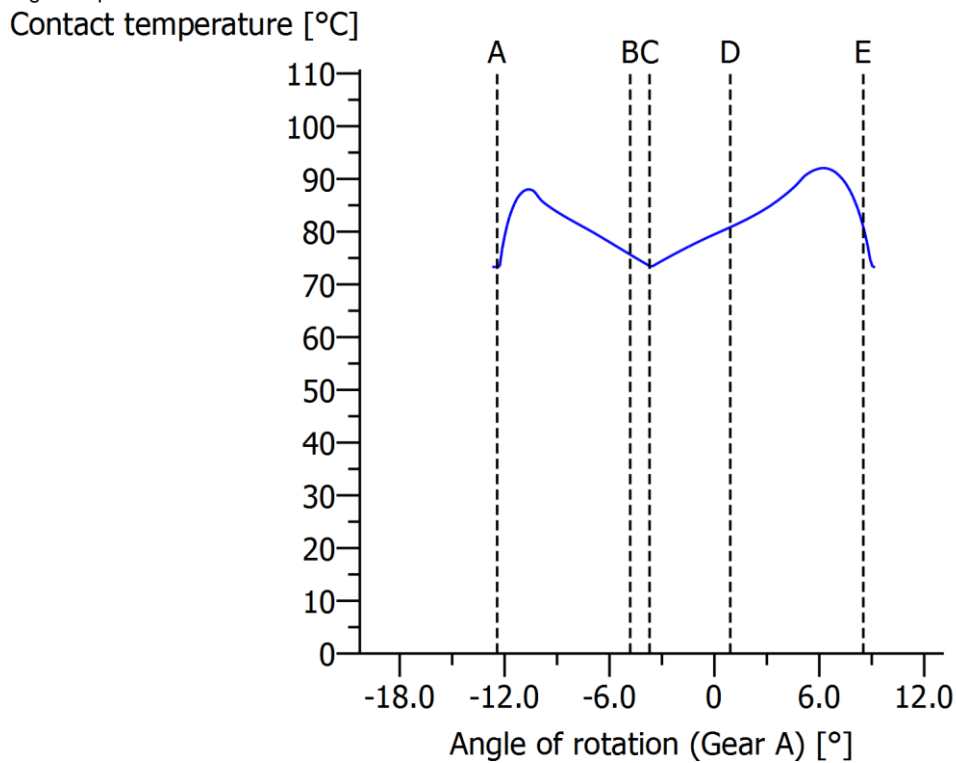


wt = 100 %, a = 101.845 mm, fpt = 0.000 μm, μ = 0.107 Working flank: Right flank Maximum sliding velocity: 1.790 m/s

Figure: Kinematics

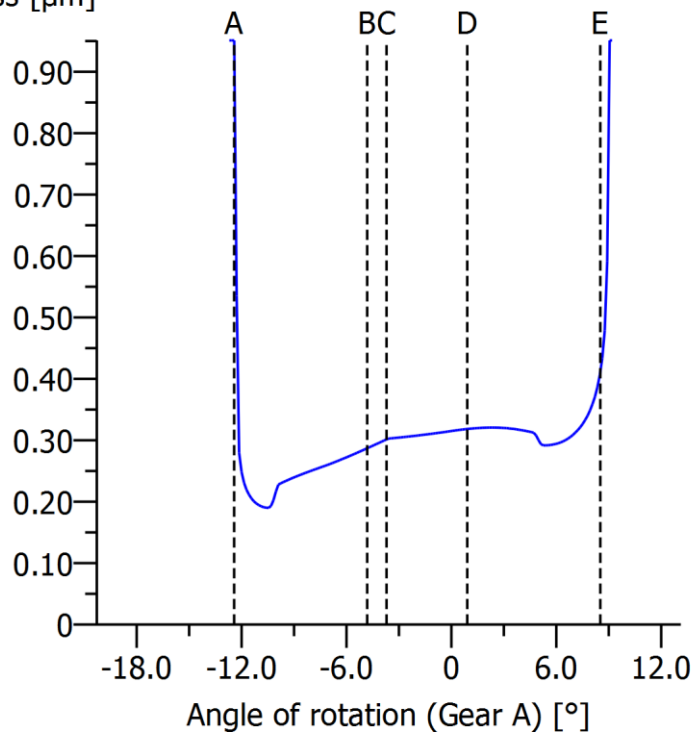


wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$  Displaying power losses per mm facewidth Working flank: Right flank  
 Figure: Specific Power Loss



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$  theoil = 70.0 °C, theM = 73.3 °C, etaM = 28.55 mPa\*s Working flank:  
 Right flank  
 Figure: Contact temperature

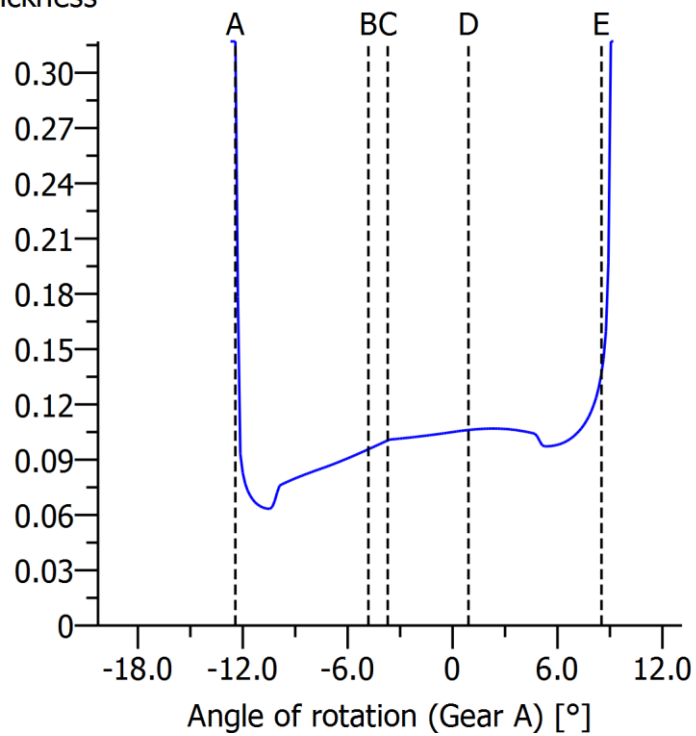
Film thickness [ $\mu\text{m}$ ]



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$ ,  $\theta_{oil} = 70.0$   $^{\circ}\text{C}$ ,  $\theta_{M} = 73.3$   $^{\circ}\text{C}$ ,  $\eta_{M} = 28.55$  mPa $\cdot$ s,  $\lambda_{Mini}(\text{ISO}) = 0.141$   $\mu\text{m}$ , Ra = 3.000  $\mu\text{m}$  Working flank: Right flank

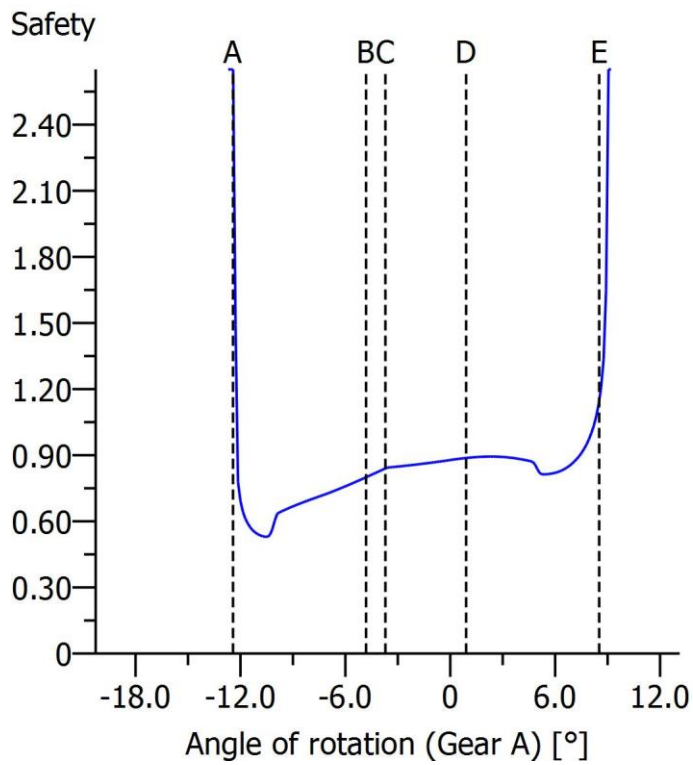
Figure: Lubricating film (ISO TR 15144)

Specific film thickness



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$ ,  $\theta_{oil} = 70.0$   $^{\circ}\text{C}$ ,  $\theta_{M} = 73.3$   $^{\circ}\text{C}$ ,  $\eta_{M} = 28.55$  mPa $\cdot$ s,  $\lambda_{Mini}(\text{ISO}) = 0.141$   $\mu\text{m}$ , Ra = 3.000  $\mu\text{m}$ ,  $\lambda_{GFmin} = 0.047$  Working flank: Right flank

Figure: Specific film thickness (ISO TR 15144)



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.107$  the oil = 70.0 °C, theM = 73.3 °C, etaM = 28.55 mPa\*s Slam(ISO) = 0.392,  
 Ra = 3.000  $\mu\text{m}$  Working flank: Right flank  
 Figure: Safety against micropitting (ISO TR 15144)

## 3.2 Report of contact analysis of the end variant with main load stage

KISSsoft Release 03/2015 D

KISSsoft-Entwicklungs-Version KISSsoft AG CH-8608 BUBIKON  
File

Filename: C:/KISSsoft 03-2015/example/CylGearPair 6a (shafts for contact analysis) (Helical DIN3990).Z12  
Description: KISSsoft example  
Changed by: mhoffmann am: 31.07.2015 um: 13:55:53

### Contact Analysis

Meshing gear 1 - gear 2

Accuracy of calculation

Medium

Partial load for calculation  $[w_t]$  100.0000 (%)

Note: In order to obtain contact analysis results for scuffing, micropitting and tooth flank fracture according to method A of ISO the calculation would have to be performed with  $w_t = 100 \cdot K_{gam} \cdot K_A \cdot K_V$ .

$(w_t \cdot K_{gam} \cdot K_A \cdot K_V = 107.52 \%)$

Working flank

Right tooth flank

Center distance	[a]	101.8450	(mm)
Single pitch deviation	$[f_{pt}]$	0.0000	( $\mu\text{m}$ )
Coefficient. of friction	$[\mu]$	0.0973	
Deviation error of axis	$[f_{\Sigma\beta}]$	0.0000	( $\mu\text{m}$ )
Inclination error of axis	$[f_{\Sigma\delta}]$	0.0000	( $\mu\text{m}$ )
Torque	$[T_1]$	40.4631	(Nm)

Torsion

from shaft calculation

		min	max	$\Delta$	$\mu$	$\sigma$
Transmission error	( $\mu\text{m}$ )	-6.1094	-6.0009	0.1085	-6.0577	0.0385
Tangent Stiffness curve	(N/ $\mu\text{m}$ )	368.2074	378.7227	10.5154	374.3494	3.3633
Secant stiffness curve	(N/ $\mu\text{m}$ )	335.1637	341.0886	5.9249	337.9938	2.0914
Line load	(N/mm)	0.0000	96.6901	96.6901	65.8095	20.9818
Torque Gear 1	(Nm)	40.4594	40.4670	0.0077	40.4636	0.0020
Torque Gear 2	(Nm)	154.0399	154.1414	0.1015	154.1006	0.0283
Power loss	(W)	138.6316	147.7028	9.0712	143.7230	2.8966
Contact temperature	( $^{\circ}\text{C}$ )	73.0861	95.1311	22.0451	82.0280	5.5623
Thickness of lubrication film	( $\mu\text{m}$ )	0.1709	1.2579	1.0870	0.3279	0.1946
Hertzian pressure	(N/mm <sup>2</sup> )		1046.4922		618.9214	
Tooth root stress gear 1	(N/mm <sup>2</sup> )		206.9704		144.8776	
Tooth root stress gear 2	(N/mm <sup>2</sup> )		167.4844		142.4823	

Safety against micropitting (ISO/TR 15144 Method A) 0.4763

Safety against scuffing 18.0208

Transverse contact ratio under load  $[\epsilon\alpha]$  1.6632

min 1.6526

$\mu$  1.6533

max 1.6632

Overlap ratio under load  $[\epsilon\beta]$  0.9847

Total contact ratio under load  $[\epsilon\gamma]$  2.6478

Efficiency  $[\eta]$  98.8500

Face load factor (ISO 6336-1 Appendix E)

$[K_{H\beta}]$  1.0023

Note: The resulting safeties do not correspond with Method A according ISO because  $K_Y$ ,  $K_A$  and  $K_V$  are not taken into account.

Amplitude spectrum of the transmission error

Harmonics	Amplitude (μm)
1	0.052
2	0.011
3	0.010
4	0.001
5	0.003
6	0.002
7	0.000
8	0.001
9	0.001
10	0.001

**K<sub>Hβ</sub> Calculation - Gear 1 - Gear 2**

Right flank

Shaft file A: C:/KISSsoft 03-2015/example/Pinion shaft for CylGearPair 6a.W10, selected gear: Pinion from CylGearPair6 (gears as masses and stiffness)

Shaft file B: C:/KISSsoft 03-2015/example/Gear shaft for CylGearPair 6a.W10, selected gear: Gear from CylGearPair6 (gears as masses and stiffness)

(Partial load for calculation  $w_t = 100\%$ )

$f_{ma} = 0.000\ \mu\text{m}$  ,  $f_{H\beta} = 0.000\ \mu\text{m}$

Result after  $i = 2$  iterations of load distribution

Gear 1

Point in polar co-ordinates:

$R = 20.991\ \text{mm}$  ,  $\varphi = 0.000^\circ$

Displacement calculated in direction  $111.425^\circ$

	y	$\varphi_{1.t}$	f1.t	f1.b	f1.tot	f1.C	f1.tot+f1.C
1	40.196 mm	0.0000°	-0.0045 μm	-0.0445 μm	-0.0490 μm	0.0000 μm	-0.0490 μm
2	40.588 mm	-0.0000°	0.0094 μm	0.0935 μm	0.1029 μm	-0.1515 μm	-0.0486 μm
3	40.980 mm	-0.0001°	0.0233 μm	0.2315 μm	0.2548 μm	-0.3030 μm	-0.0482 μm
4	41.373 mm	-0.0001°	0.0368 μm	0.3691 μm	0.4059 μm	-0.4535 μm	-0.0476 μm
5	41.765 mm	-0.0001°	0.0502 μm	0.5065 μm	0.5567 μm	-0.6038 μm	-0.0471 μm
6	42.157 mm	-0.0002°	0.0632 μm	0.6437 μm	0.7069 μm	-0.7534 μm	-0.0465 μm
7	42.549 mm	-0.0002°	0.0759 μm	0.7805 μm	0.8564 μm	-0.9024 μm	-0.0460 μm
8	42.941 mm	-0.0003°	0.0884 μm	0.9172 μm	1.0056 μm	-1.0511 μm	-0.0455 μm
9	43.333 mm	-0.0003°	0.1005 μm	1.0534 μm	1.1539 μm	-1.1989 μm	-0.0450 μm
10	43.725 mm	-0.0003°	0.1123 μm	1.1895 μm	1.3018 μm	-1.3463 μm	-0.0445 μm
11	44.118 mm	-0.0004°	0.1239 μm	1.3253 μm	1.4492 μm	-1.4932 μm	-0.0440 μm
12	44.510 mm	-0.0004°	0.1351 μm	1.4607 μm	1.5958 μm	-1.6394 μm	-0.0436 μm
13	44.902 mm	-0.0004°	0.1462 μm	1.5960 μm	1.7421 μm	-1.7852 μm	-0.0431 μm
14	45.294 mm	-0.0005°	0.1568 μm	1.7308 μm	1.8876 μm	-1.9303 μm	-0.0427 μm
15	45.686 mm	-0.0005°	0.1671 μm	1.8654 μm	2.0326 μm	-2.0749 μm	-0.0424 μm
16	46.078 mm	-0.0005°	0.1773 μm	1.9998 μm	2.1771 μm	-2.2191 μm	-0.0420 μm
17	46.471 mm	-0.0005°	0.1870 μm	2.1338 μm	2.3208 μm	-2.3624 μm	-0.0417 μm
18	46.863 mm	-0.0006°	0.1965 μm	2.2676 μm	2.4641 μm	-2.5054 μm	-0.0413 μm
19	47.255 mm	-0.0006°	0.2057 μm	2.4010 μm	2.6067 μm	-2.6478 μm	-0.0411 μm
20	47.647 mm	-0.0006°	0.2145 μm	2.5341 μm	2.7487 μm	-2.7896 μm	-0.0409 μm
21	48.039 mm	-0.0007°	0.2232 μm	2.6671 μm	2.8903 μm	-2.9310 μm	-0.0407 μm
22	48.431 mm	-0.0007°	0.2314 μm	2.7996 μm	3.0310 μm	-3.0715 μm	-0.0405 μm
23	48.824 mm	-0.0007°	0.2395 μm	2.9319 μm	3.1713 μm	-3.2117 μm	-0.0404 μm
24	49.216 mm	-0.0007°	0.2472 μm	3.0639 μm	3.3110 μm	-3.3513 μm	-0.0403 μm
25	49.608 mm	-0.0007°	0.2545 μm	3.1955 μm	3.4500 μm	-3.4903 μm	-0.0402 μm
26	50.000 mm	-0.0008°	0.2617 μm	3.3270 μm	3.5887 μm	-3.6289 μm	-0.0402 μm
27	50.392 mm	-0.0008°	0.2684 μm	3.4580 μm	3.7264 μm	-3.7666 μm	-0.0402 μm

28	50.784 mm	-0.0008°	0.2750 μm	3.5888 μm	3.8638 μm	-3.9040 μm	-0.0403 μm
29	51.176 mm	-0.0008°	0.2812 μm	3.7192 μm	4.0005 μm	-4.0408 μm	-0.0404 μm
30	51.569 mm	-0.0008°	0.2871 μm	3.8494 μm	4.1365 μm	-4.1770 μm	-0.0405 μm
31	51.961 mm	-0.0009°	0.2928 μm	3.9793 μm	4.2721 μm	-4.3128 μm	-0.0407 μm
32	52.353 mm	-0.0009°	0.2981 μm	4.1088 μm	4.4069 μm	-4.4478 μm	-0.0409 μm
33	52.745 mm	-0.0009°	0.3031 μm	4.2381 μm	4.5412 μm	-4.5824 μm	-0.0412 μm
34	53.137 mm	-0.0009°	0.3079 μm	4.3671 μm	4.6750 μm	-4.7165 μm	-0.0415 μm
35	53.529 mm	-0.0009°	0.3123 μm	4.4957 μm	4.8080 μm	-4.8498 μm	-0.0418 μm
36	53.922 mm	-0.0009°	0.3165 μm	4.6241 μm	4.9406 μm	-4.9828 μm	-0.0422 μm
37	54.314 mm	-0.0009°	0.3203 μm	4.7521 μm	5.0724 μm	-5.1151 μm	-0.0427 μm
38	54.706 mm	-0.0009°	0.3239 μm	4.8798 μm	5.2037 μm	-5.2468 μm	-0.0431 μm
39	55.098 mm	-0.0010°	0.3272 μm	5.0073 μm	5.3345 μm	-5.3781 μm	-0.0436 μm
40	55.490 mm	-0.0010°	0.3301 μm	5.1344 μm	5.4645 μm	-5.5087 μm	-0.0442 μm
41	55.882 mm	-0.0010°	0.3328 μm	5.2613 μm	5.5941 μm	-5.6389 μm	-0.0448 μm
42	56.275 mm	-0.0010°	0.3351 μm	5.3878 μm	5.7229 μm	-5.7683 μm	-0.0454 μm
43	56.667 mm	-0.0010°	0.3372 μm	5.5140 μm	5.8512 μm	-5.8973 μm	-0.0461 μm
44	57.059 mm	-0.0010°	0.3390 μm	5.6400 μm	5.9791 μm	-6.0259 μm	-0.0468 μm
45	57.451 mm	-0.0010°	0.3405 μm	5.7656 μm	6.1060 μm	-6.1536 μm	-0.0476 μm
46	57.843 mm	-0.0010°	0.3417 μm	5.8910 μm	6.2326 μm	-6.2810 μm	-0.0483 μm
47	58.235 mm	-0.0010°	0.3425 μm	6.0160 μm	6.3585 μm	-6.4077 μm	-0.0492 μm
48	58.627 mm	-0.0010°	0.3431 μm	6.1407 μm	6.4838 μm	-6.5338 μm	-0.0500 μm
49	59.020 mm	-0.0010°	0.3435 μm	6.2652 μm	6.6087 μm	-6.6596 μm	-0.0509 μm
50	59.412 mm	-0.0010°	0.3436 μm	6.3893 μm	6.7329 μm	-6.7846 μm	-0.0517 μm
51	59.804 mm	-0.0010°	0.3436 μm	6.5134 μm	6.8570 μm	-6.9094 μm	-0.0525 μm

#### Gear 2

Point in polar co-ordinates:

R = 80.854 mm ,  $\varphi = 180.000^\circ$

Displacement calculated in direction 111.425 °

	y	$\varphi_{2.t}$	f2.t	f2.b	f2.tot	f2.C	f2.tot+f2.C
1	24.196 mm	-0.0000°	-0.0000 μm	0.0230 μm	0.0230 μm	0.0000 μm	0.0230 μm
2	24.588 mm	0.0000°	0.0000 μm	-0.0484 μm	-0.0483 μm	0.0706 μm	0.0222 μm
3	24.980 mm	0.0000°	0.0000 μm	-0.1197 μm	-0.1197 μm	0.1412 μm	0.0215 μm
4	25.373 mm	0.0000°	0.0001 μm	-0.1911 μm	-0.1910 μm	0.2118 μm	0.0208 μm
5	25.765 mm	0.0000°	0.0001 μm	-0.2624 μm	-0.2623 μm	0.2824 μm	0.0200 μm
6	26.157 mm	0.0000°	0.0002 μm	-0.3338 μm	-0.3336 μm	0.3529 μm	0.0194 μm
7	26.549 mm	0.0000°	0.0003 μm	-0.4051 μm	-0.4048 μm	0.4235 μm	0.0187 μm
8	26.941 mm	0.0000°	0.0004 μm	-0.4764 μm	-0.4760 μm	0.4941 μm	0.0181 μm
9	27.333 mm	0.0000°	0.0005 μm	-0.5477 μm	-0.5472 μm	0.5647 μm	0.0175 μm
10	27.725 mm	0.0000°	0.0007 μm	-0.6190 μm	-0.6183 μm	0.6353 μm	0.0170 μm
11	28.118 mm	0.0000°	0.0009 μm	-0.6903 μm	-0.6894 μm	0.7059 μm	0.0165 μm
12	28.510 mm	0.0000°	0.0010 μm	-0.7615 μm	-0.7605 μm	0.7765 μm	0.0160 μm
13	28.902 mm	0.0000°	0.0012 μm	-0.8328 μm	-0.8315 μm	0.8471 μm	0.0155 μm
14	29.294 mm	0.0000°	0.0015 μm	-0.9040 μm	-0.9026 μm	0.9176 μm	0.0151 μm
15	29.686 mm	0.0000°	0.0017 μm	-0.9752 μm	-0.9735 μm	0.9882 μm	0.0147 μm
16	30.078 mm	0.0000°	0.0020 μm	-1.0465 μm	-1.0445 μm	1.0588 μm	0.0143 μm
17	30.471 mm	0.0000°	0.0022 μm	-1.1177 μm	-1.1154 μm	1.1294 μm	0.0140 μm
18	30.863 mm	0.0000°	0.0025 μm	-1.1889 μm	-1.1863 μm	1.2000 μm	0.0137 μm
19	31.255 mm	0.0000°	0.0028 μm	-1.2600 μm	-1.2572 μm	1.2706 μm	0.0134 μm
20	31.647 mm	0.0000°	0.0032 μm	-1.3312 μm	-1.3280 μm	1.3412 μm	0.0131 μm
21	32.039 mm	0.0000°	0.0035 μm	-1.4023 μm	-1.3988 μm	1.4118 μm	0.0129 μm
22	32.431 mm	0.0000°	0.0039 μm	-1.4735 μm	-1.4696 μm	1.4824 μm	0.0127 μm
23	32.824 mm	0.0000°	0.0043 μm	-1.5446 μm	-1.5404 μm	1.5529 μm	0.0126 μm
24	33.216 mm	0.0000°	0.0047 μm	-1.6157 μm	-1.6111 μm	1.6235 μm	0.0125 μm
25	33.608 mm	0.0000°	0.0051 μm	-1.6868 μm	-1.6817 μm	1.6941 μm	0.0124 μm
26	34.000 mm	0.0000°	0.0055 μm	-1.7579 μm	-1.7524 μm	1.7647 μm	0.0123 μm
27	34.392 mm	0.0000°	0.0060 μm	-1.8290 μm	-1.8230 μm	1.8353 μm	0.0123 μm

28	34.784 mm	0.0000°	0.0065 µm	-1.9001 µm	-1.8936 µm	1.9059 µm	0.0123 µm
29	35.176 mm	0.0000°	0.0069 µm	-1.9711 µm	-1.9642 µm	1.9765 µm	0.0123 µm
30	35.569 mm	0.0000°	0.0075 µm	-2.0421 µm	-2.0347 µm	2.0471 µm	0.0124 µm
31	35.961 mm	0.0000°	0.0080 µm	-2.1132 µm	-2.1052 µm	2.1176 µm	0.0125 µm
32	36.353 mm	0.0000°	0.0085 µm	-2.1842 µm	-2.1756 µm	2.1882 µm	0.0126 µm
33	36.745 mm	0.0000°	0.0091 µm	-2.2552 µm	-2.2461 µm	2.2588 µm	0.0127 µm
34	37.137 mm	0.0000°	0.0097 µm	-2.3262 µm	-2.3165 µm	2.3294 µm	0.0129 µm
35	37.529 mm	0.0000°	0.0103 µm	-2.3971 µm	-2.3869 µm	2.4000 µm	0.0131 µm
36	37.922 mm	0.0000°	0.0109 µm	-2.4681 µm	-2.4572 µm	2.4706 µm	0.0134 µm
37	38.314 mm	0.0000°	0.0115 µm	-2.5390 µm	-2.5275 µm	2.5412 µm	0.0137 µm
38	38.706 mm	0.0000°	0.0122 µm	-2.6100 µm	-2.5978 µm	2.6118 µm	0.0140 µm
39	39.098 mm	0.0000°	0.0129 µm	-2.6809 µm	-2.6680 µm	2.6824 µm	0.0143 µm
40	39.490 mm	0.0000°	0.0136 µm	-2.7518 µm	-2.7383 µm	2.7529 µm	0.0147 µm
41	39.882 mm	0.0000°	0.0143 µm	-2.8227 µm	-2.8085 µm	2.8235 µm	0.0151 µm
42	40.275 mm	0.0000°	0.0150 µm	-2.8936 µm	-2.8786 µm	2.8941 µm	0.0155 µm
43	40.667 mm	0.0000°	0.0157 µm	-2.9645 µm	-2.9487 µm	2.9647 µm	0.0160 µm
44	41.059 mm	0.0000°	0.0165 µm	-3.0353 µm	-3.0188 µm	3.0353 µm	0.0165 µm
45	41.451 mm	0.0000°	0.0173 µm	-3.1062 µm	-3.0889 µm	3.1059 µm	0.0170 µm
46	41.843 mm	0.0000°	0.0181 µm	-3.1770 µm	-3.1589 µm	3.1765 µm	0.0176 µm
47	42.235 mm	0.0000°	0.0189 µm	-3.2478 µm	-3.2289 µm	3.2471 µm	0.0181 µm
48	42.627 mm	0.0000°	0.0197 µm	-3.3186 µm	-3.2989 µm	3.3176 µm	0.0188 µm
49	43.020 mm	0.0000°	0.0206 µm	-3.3894 µm	-3.3688 µm	3.3882 µm	0.0194 µm
50	43.412 mm	0.0000°	0.0215 µm	-3.4602 µm	-3.4387 µm	3.4588 µm	0.0201 µm
51	43.804 mm	0.0000°	0.0223 µm	-3.5310 µm	-3.5086 µm	3.5294 µm	0.0208 µm

Explanations:

y	: Width
φ.t	: Static torsion
f.t	: Displacement due to torsion
f.b	: Displacement due to bending
f.tot	: Total displacement (f.b+f.t)
f.C	: Change due to flank line modification

Load distribution

Contact stiffness = 18.895 N/mm/µm

Young's modulus = 206000.0/206000.0 N/mm<sup>2</sup>

	y	δ	g	w
1.	40.1961 mm	0.0012 µm	5.9378 µm	112.1957 N/mm
2.	40.5882 mm	0.0024 µm	5.9366 µm	112.1737 N/mm
3.	40.9804 mm	0.0036 µm	5.9354 µm	112.1516 N/mm
4.	41.3725 mm	0.0048 µm	5.9341 µm	112.1273 N/mm
5.	41.7647 mm	0.0061 µm	5.9329 µm	112.1034 N/mm
6.	42.1569 mm	0.0073 µm	5.9317 µm	112.0803 N/mm
7.	42.5490 mm	0.0085 µm	5.9305 µm	112.0582 N/mm
8.	42.9412 mm	0.0096 µm	5.9294 µm	112.0366 N/mm
9.	43.3333 mm	0.0107 µm	5.9283 µm	112.0163 N/mm
10.	43.7255 mm	0.0118 µm	5.9272 µm	111.9966 N/mm
11.	44.1176 mm	0.0128 µm	5.9262 µm	111.9777 N/mm
12.	44.5098 mm	0.0137 µm	5.9253 µm	111.9597 N/mm
13.	44.9020 mm	0.0146 µm	5.9244 µm	111.9423 N/mm
14.	45.2941 mm	0.0154 µm	5.9236 µm	111.9273 N/mm
15.	45.6863 mm	0.0162 µm	5.9228 µm	111.9127 N/mm
16.	46.0784 mm	0.0170 µm	5.9220 µm	111.8985 N/mm
17.	46.4706 mm	0.0176 µm	5.9214 µm	111.8861 N/mm
18.	46.8627 mm	0.0182 µm	5.9208 µm	111.8744 N/mm
19.	47.2549 mm	0.0187 µm	5.9202 µm	111.8646 N/mm
20.	47.6471 mm	0.0192 µm	5.9198 µm	111.8557 N/mm
21.	48.0392 mm	0.0196 µm	5.9193 µm	111.8476 N/mm



22.	48.4314 mm	0.0200 μm	5.9190 μm	111.8412 N/mm
23.	48.8235 mm	0.0203 μm	5.9187 μm	111.8355 N/mm
24.	49.2157 mm	0.0205 μm	5.9185 μm	111.8312 N/mm
25.	49.6078 mm	0.0206 μm	5.9184 μm	111.8288 N/mm
26.	50.0000 mm	0.0207 μm	5.9183 μm	111.8274 N/mm
27.	50.3922 mm	0.0207 μm	5.9183 μm	111.8271 N/mm
28.	50.7843 mm	0.0207 μm	5.9183 μm	111.8279 N/mm
29.	51.1765 mm	0.0206 μm	5.9184 μm	111.8305 N/mm
30.	51.5686 mm	0.0203 μm	5.9187 μm	111.8346 N/mm
31.	51.9608 mm	0.0201 μm	5.9189 μm	111.8395 N/mm
32.	52.3529 mm	0.0197 μm	5.9193 μm	111.8464 N/mm
33.	52.7451 mm	0.0193 μm	5.9197 μm	111.8541 N/mm
34.	53.1373 mm	0.0188 μm	5.9202 μm	111.8629 N/mm
35.	53.5294 mm	0.0183 μm	5.9207 μm	111.8738 N/mm
36.	53.9216 mm	0.0176 μm	5.9214 μm	111.8856 N/mm
37.	54.3137 mm	0.0169 μm	5.9221 μm	111.8996 N/mm
38.	54.7059 mm	0.0161 μm	5.9229 μm	111.9142 N/mm
39.	55.0980 mm	0.0153 μm	5.9237 μm	111.9294 N/mm
40.	55.4902 mm	0.0144 μm	5.9246 μm	111.9474 N/mm
41.	55.8824 mm	0.0134 μm	5.9256 μm	111.9664 N/mm
42.	56.2745 mm	0.0123 μm	5.9267 μm	111.9866 N/mm
43.	56.6667 mm	0.0112 μm	5.9278 μm	112.0081 N/mm
44.	57.0588 mm	0.0100 μm	5.9290 μm	112.0306 N/mm
45.	57.4510 mm	0.0087 μm	5.9303 μm	112.0549 N/mm
46.	57.8431 mm	0.0073 μm	5.9316 μm	112.0800 N/mm
47.	58.2353 mm	0.0059 μm	5.9331 μm	112.1068 N/mm
48.	58.6275 mm	0.0044 μm	5.9346 μm	112.1349 N/mm
49.	59.0196 mm	0.0029 μm	5.9361 μm	112.1638 N/mm
50.	59.4118 mm	0.0015 μm	5.9375 μm	112.1914 N/mm
51.	59.8039 mm	0.0000 μm	5.9390 μm	112.2188 N/mm

Explanations:

δ : Gap  
g : Flank overlap  
w : Line load

Force application point, Y direction:  $y = 50.000 \text{ mm}$  ( $F = 2239.3 \text{ N}$ )

To take into account the load distribution in the shaft calculation: Force center point offset:  $\Delta y = -0.000 \text{ mm}$

$w_{\max} = 112.219 \text{ N/mm}$ ,  $w_m = 111.964 \text{ N/mm}$

$w_m = K_V * K_A * K_Y * (F_t/b) / \cos(\alpha_{wt})$

$K_V = 1.0752$ ,  $K_A = 1.000$ ,  $K_Y = 1.000$

$K_{H\beta} = w_{\max}/w_m = 1.0023$  (Calculation according to ISO 6336-1, Appendix E)

Side I, II:  $w_I / w_m = 1.0021$   $w_{II} / w_m = 1.0023$

Notice: The influence of the exceeding facewidth is not taken into account in the calculation of  $K_{H\beta}$ .

#### Calculation of the 'equivalent' linear misalignment of the shafts

IMPORTANT: The flank line modifications are taken into account.

a) Calculation of the 'equivalent' linear misalignment, so that  $K_{H\beta}$  remains the same

Linear misalignment in the plane of action, Gear 1:  $[F_{\beta x1}] 0.02 \mu\text{m}$  ( $K_{H\beta1} = 1.0014$ )

Linear misalignment in the plane of action, Gear 2:  $[F_{\beta x2}] 7.07 \mu\text{m}$  ( $K_{H\beta2} = 1.5969$ )

Axis alignment: Deviation error of axis/Inclination error of axis  $f\Sigma\beta/f\Sigma\delta: 0.03 \mu\text{m}/0.01 \mu\text{m}$

( $F_{\beta x} = 0.03 \mu\text{m}$ )

The deformation due to torsion is included in the specification of  $f\Sigma\beta/f\Sigma\delta$ .

b) Linear misalignment due to connection of the gap from side I to II

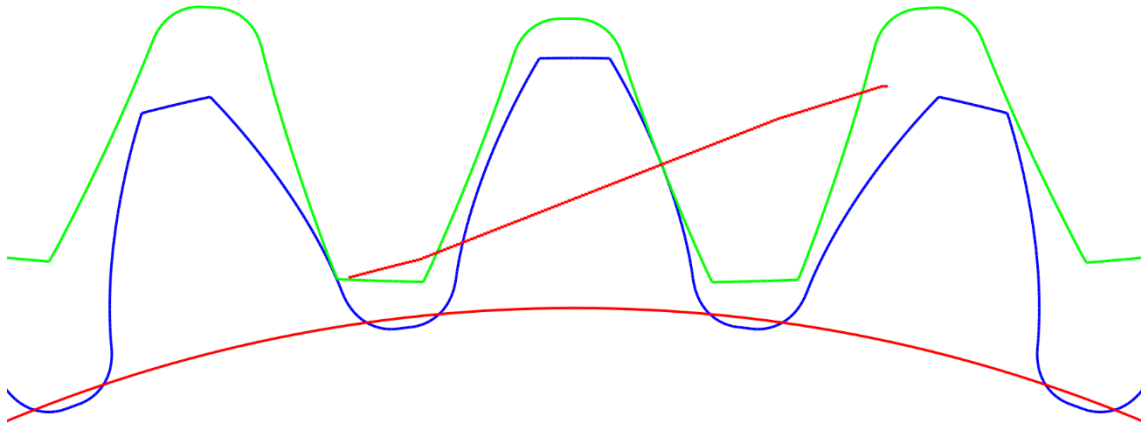
Linear misalignment in the plane of action, Gear 1:  $[F_{\beta x1}] 0.00 \mu\text{m}$

Linear misalignment in the plane of action, Gear 2:  $[F_{\beta x2}]$  7.06  $\mu\text{m}$

Axis alignment: Deviation error of axis/Inclination error of axis  $f_{\Sigma\beta}/f_{\Sigma\delta}$ : -0.00  $\mu\text{m}/-0.00 \mu\text{m}$

( $F_{\beta x} = -0.00 \mu\text{m}$ )

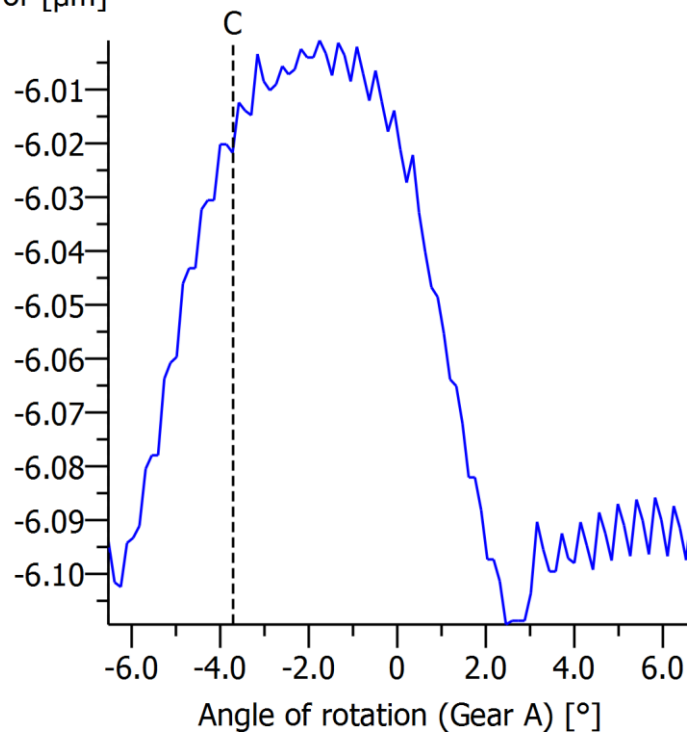
The deformation due to torsion is included in the specification of  $f_{\Sigma\beta}/f_{\Sigma\delta}$ .



$da1 = 45.6874 \text{ mm}$ ,  $df1 = 38.8180 \text{ mm}$ ,  $As1 = -0.0540 \text{ mm}$ ,  $da2 = 163.9684 \text{ mm}$ ,  $df2 = 156.9589 \text{ mm}$ ,  $As2 = -0.1050 \text{ mm}$

Figure: Path of contact

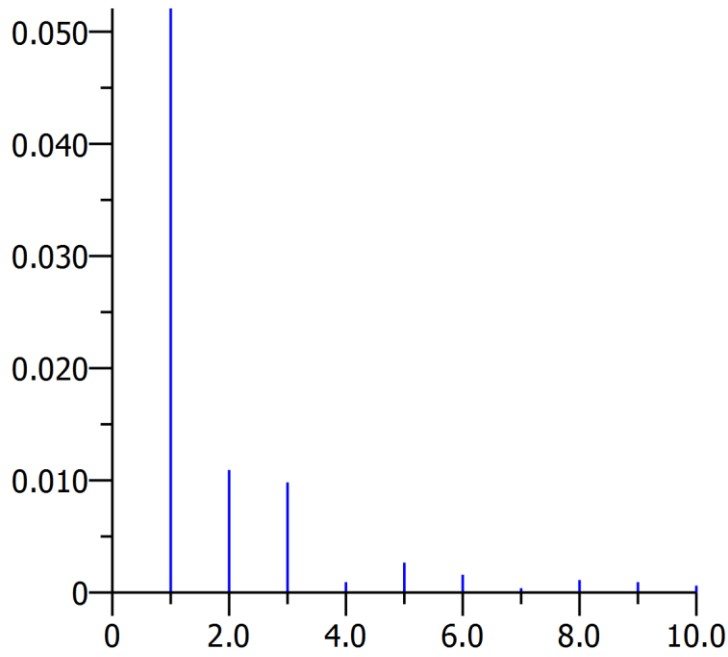
Transmission error [ $\mu\text{m}$ ]



$wt = 100 \%$ ,  $a = 101.845 \text{ mm}$ ,  $f_{pt} = 0.000 \mu\text{m}$ ,  $\mu = 0.097$  Working flank: Right flank

Figure: Transmission error

Amplitude [ $\mu\text{m}$ ]

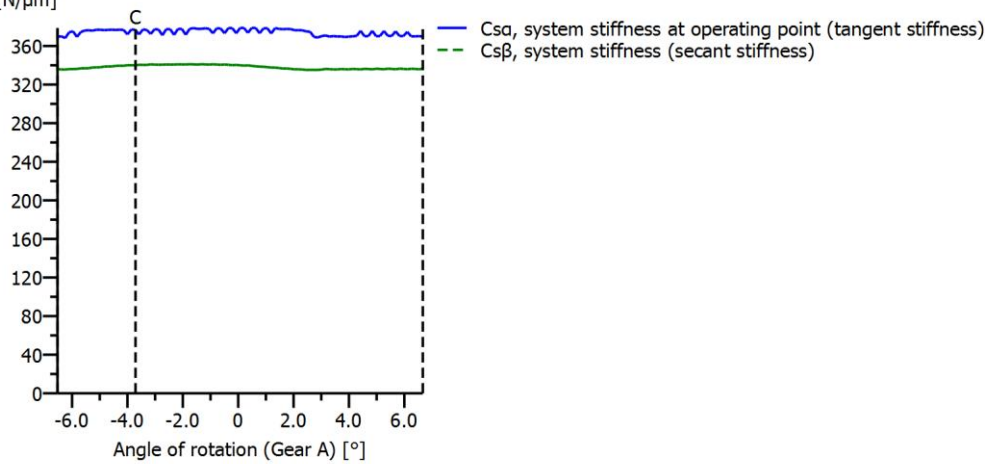


Arrangement of harmonics

Arrangement of harmonics	Amplitude [ $\mu\text{m}$ ]	1.	0.0520762.	0.0109263.	0.0098204.	0.0009285.
	0.0026666.	0.0015867.	0.0003898.	0.0011189.	0.00092910.	0.000615

wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu$  = 0.097 Working flank: Right flank

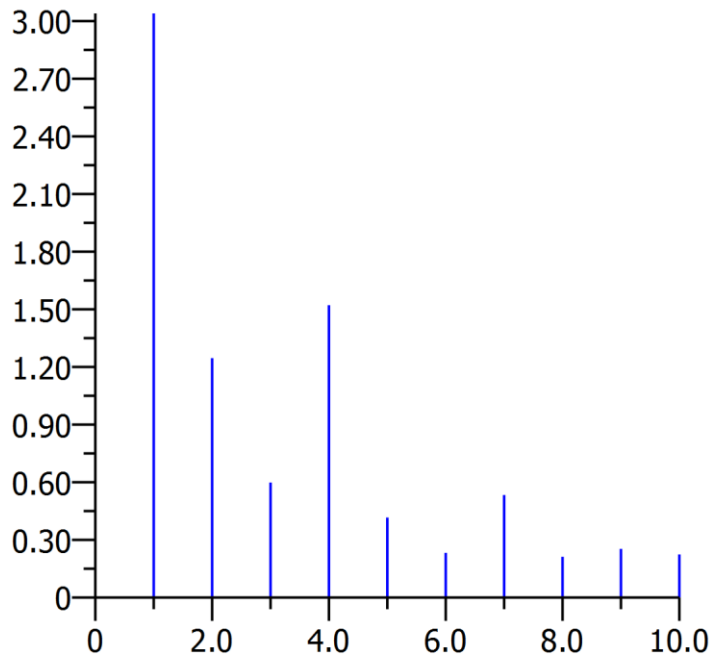
Figure: Amplitude spectrum of transmission error  
System stiffness [ $\text{N}/\mu\text{m}$ ]



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu$  = 0.097 Working flank: Right flank  
 $C_{s\alpha\_mean} = 374.3984863 \text{ N}/\mu\text{m}$   
 $C_{s\beta\_mean} = 338.0123363 \text{ N}/\mu\text{m}$   
 $C_{s\alpha} = C_{y\alpha} * b$   
 $C_{s\beta} = C_{y\beta} * b$

Figure: Stiffness curve

Amplitude [ $\mu\text{m}$ ]



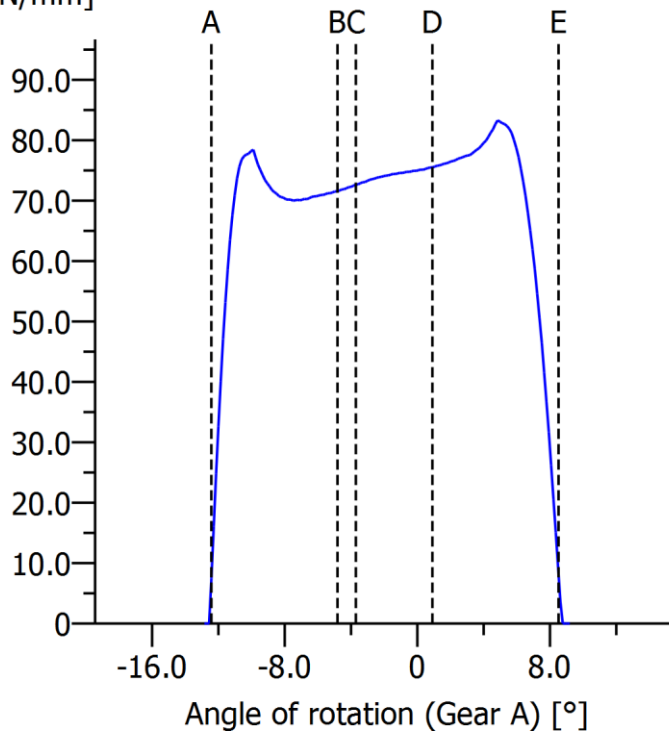
Arrangement of harmonics

Arrangement of harmonics	Amplitude [N/mm/ $\mu\text{m}$ ]	1.	3.0399272.	1.2459713.	0.5984794.
1.5214215.	0.4163106.	0.2324517.	0.5334218.	0.2120979.	0.25316210.

0.224151 wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu$  = 0.097 Working flank: Right flank

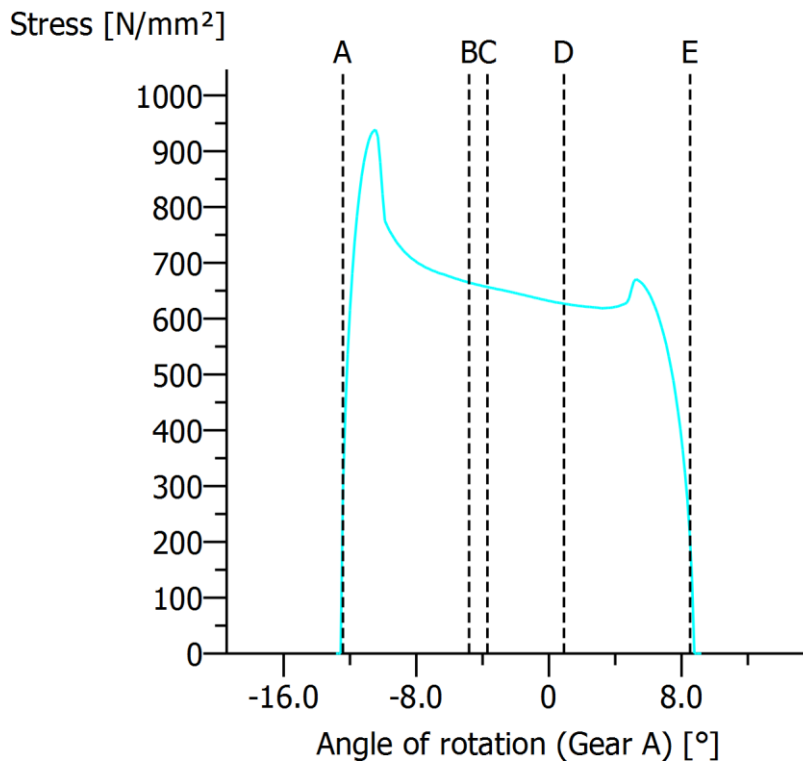
Figure: Amplitude spectrum of contact stiffness

Normal force (line load) [N/mm]



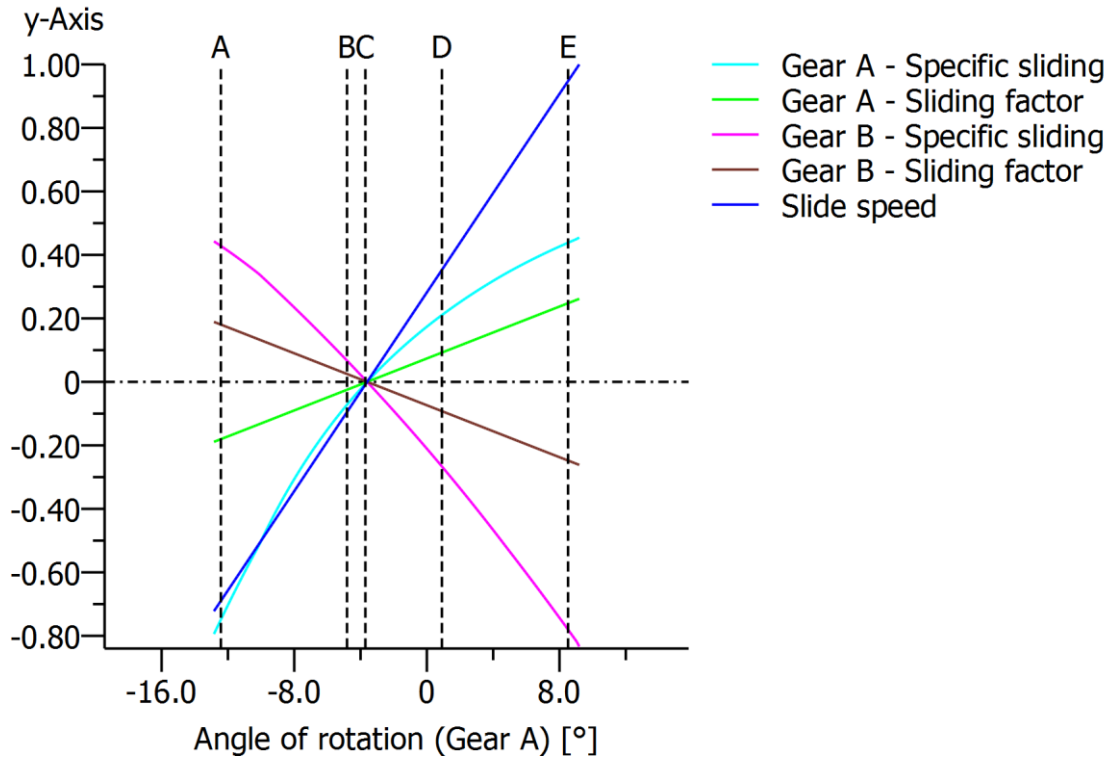
wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu$  = 0.097 Working flank: Right flank

Figure: Normal force curve (Line load)



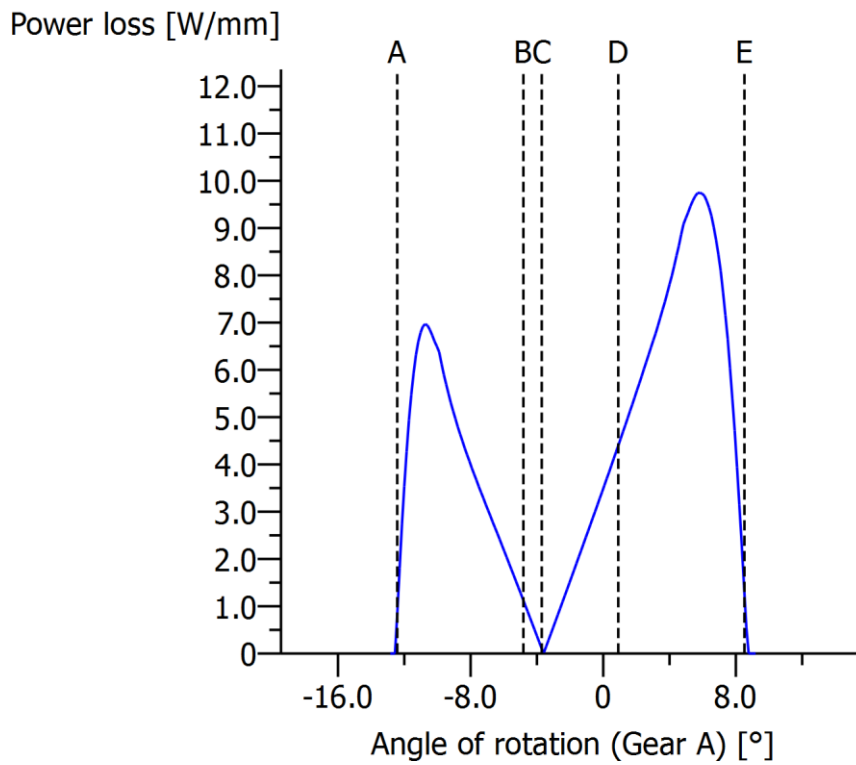
wt = 100 %, a = 101.845 mm, fpt = 0.000 μm, μ = 0.097 Working flank: Right flank

Figure: Stress curve

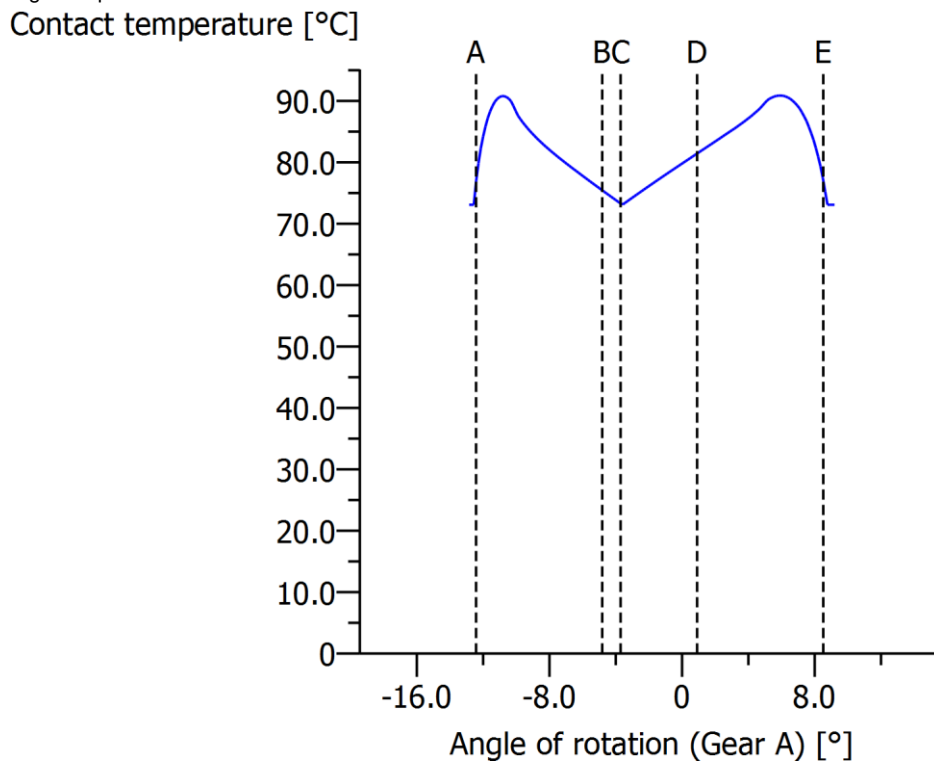


wt = 100 %, a = 101.845 mm, fpt = 0.000 μm, μ = 0.097 Working flank: Right flank Maximum sliding velocity: 1.694 m/s

Figure: Kinematics

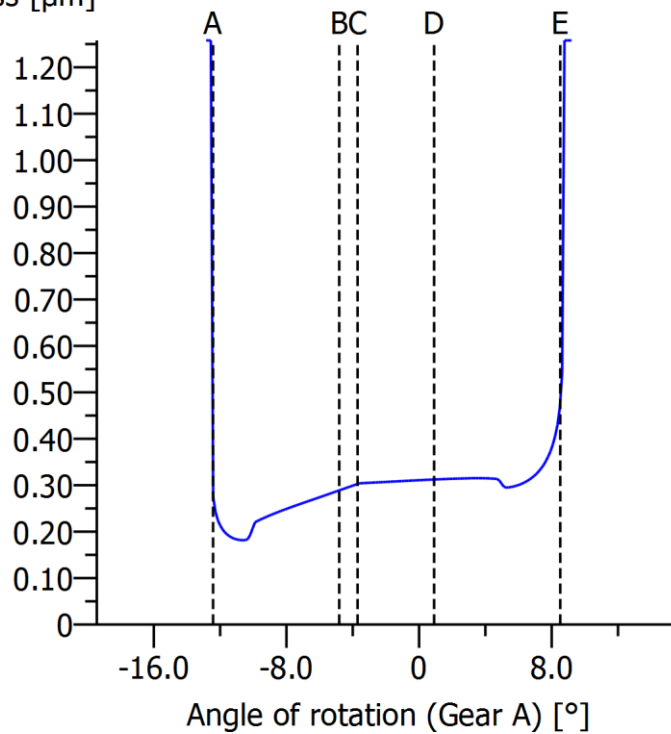


wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.097$  Displaying power losses per mm facewidth Working flank: Right flank  
 Figure: Specific Power Loss



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.097$  theoil = 70.0 °C, theM = 73.1 °C, etaM = 28.79 mPa\*s Working flank:  
 Right flank  
 Figure: Contact temperature

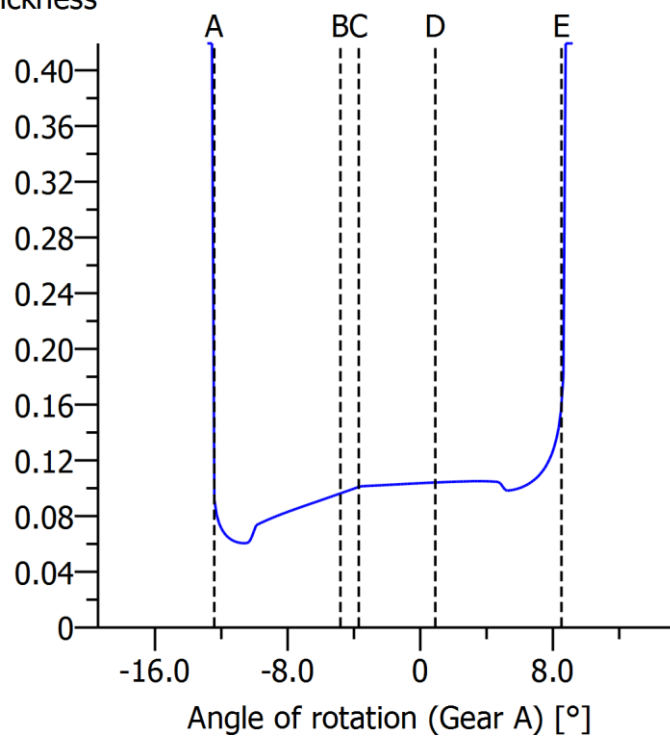
Film thickness [ $\mu\text{m}$ ]



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.097$  the oil = 70.0  $^{\circ}\text{C}$ , the M = 73.1  $^{\circ}\text{C}$ ,  $\eta_{\text{aM}} = 28.79$  mPa $\cdot$ s hMini(ISO) = 0.171  $\mu\text{m}$ , Ra = 3.000  $\mu\text{m}$  Working flank: Right flank

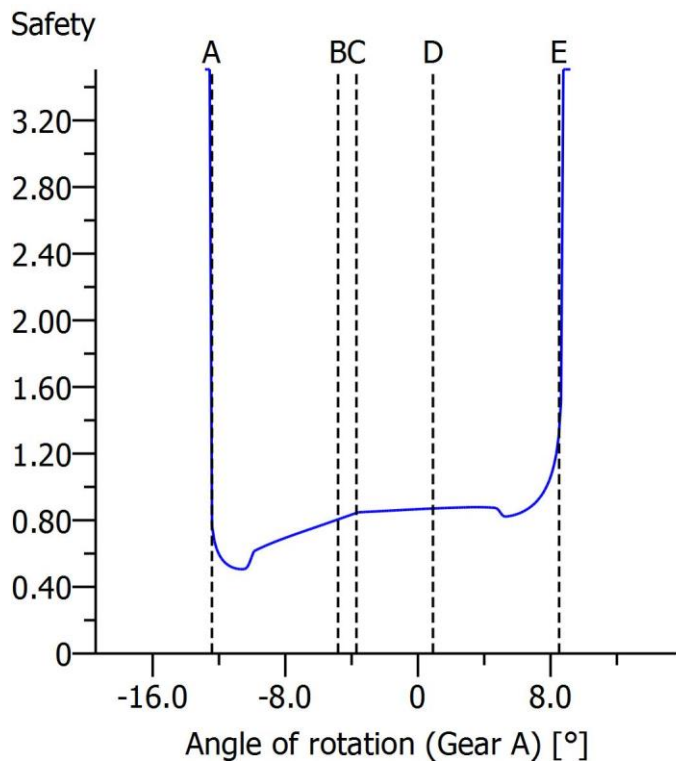
Figure: Lubricating film (ISO TR 15144)

Specific film thickness



wt = 100 %, a = 101.845 mm, fpt = 0.000  $\mu\text{m}$ ,  $\mu = 0.097$  the oil = 70.0  $^{\circ}\text{C}$ , the M = 73.1  $^{\circ}\text{C}$ ,  $\eta_{\text{aM}} = 28.79$  mPa $\cdot$ s hMini(ISO) = 0.171  $\mu\text{m}$ , Ra = 3.000  $\mu\text{m}$ ,  $\lambda_{\text{aM}} = 0.057$  Working flank: Right flank

Figure: Specific film thickness (ISO TR 15144)



wt = 100 %, a = 101.845 mm, f<sub>pt</sub> = 0.000 μm, μ = 0.097, t<sub>he0il</sub> = 70.0 °C, t<sub>heM</sub> = 73.1 °C, eta<sub>M</sub> = 28.79 mPa\*s, S<sub>lam</sub>(ISO) = 0.476, Ra = 3.000 μm Working flank: Right flank

Figure: Safety against micropitting (ISO TR 15144)